

# IEQ and Thermal Comfort: The Next Energy Efficiency Frontier?

Brian Just

# Overview

- Why we're here
- ASHRAE 55: Theory
- ASHRAE 55: Application
- Tools beyond ASHRAE
- Designing for comfort
- Beyond ASHRAE...
- Pulling it all together



# Why we're here

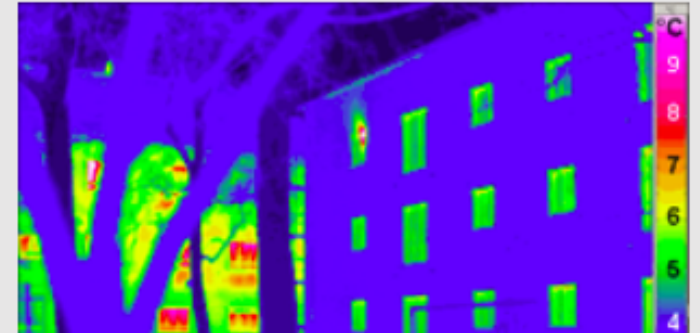
# Why we're here

## Half of Americans Surveyed Struggle to Feel Warm and Comfortable in their Homes During the Winter Months Finds RPA|Ecocor

Drafts and Poor Insulation Create Both Discomfort and Familial Strife

**MILFORD, PA AND SEARSMONT, ME (PRWEB) DECEMBER 07, 2016**

**Richard Pedranti Architect** (RPA), a full-service architecture firm focused on sustainable design, and **Ecocor**, a construction company that manufactures, delivers and assembles high performance Passive House buildings, surveyed Americans and found that regardless of how much they pay to heat their home, more than half of respondents struggle to feel comfortable in





# Why we're here

“The most common complaint facility managers hear from building occupants is that their office space is too cold.

That would seem an easy enough problem to solve, except for the fact that the number two complaint is that it's too hot.”

Ref: *Expanding the Engineers' Comfort Zone: Working with Adaptive Thermal Comfort*, BuildingGreen, 2004,  
<https://www.buildinggreen.com/feature/expanding-engineers-comfort-zone-working-adaptive-thermal-comfort>



# IEQ = Indoor Environmental Quality

$$IEQ = IAQ + ITQ + ILQ + ISQ + IOQ + IVQ$$

where I = Indoor, Q = Quality

and	A =	Air	Much more to comfort than just thermal!
	T =	Thermal	
	L =	Lighting	
	S =	Sound	
	O =	Odor	
	V =	Vibrations	



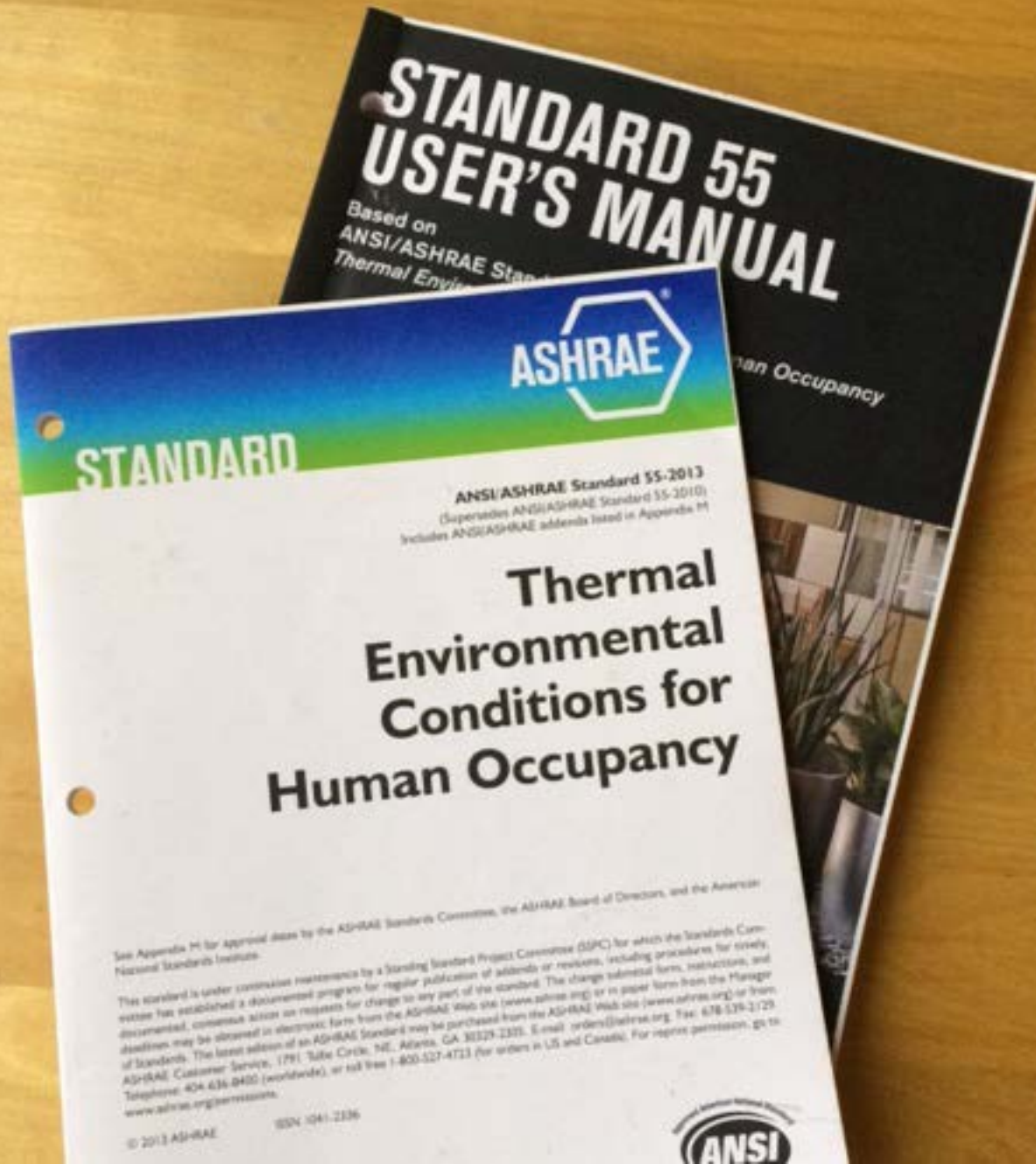
# ASHRAE 55: Theory



# ASHRAE 55

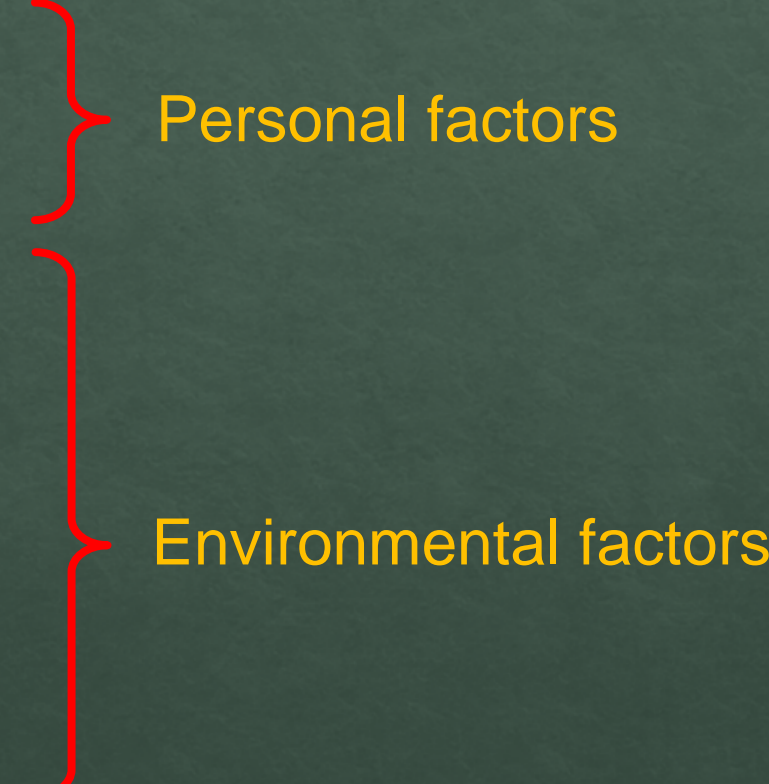
The science of comfort (sort of)

Purpose: "...to specify the combinations of indoor thermal environmental factors and personal factors that will produce thermal environmental conditions acceptable to a majority of the occupants within the space."





# Thermal comfort: 6 factors

- Metabolic rate (met)
  - Clothing insulation (clo)
  - Air temperature
  - Radiant temperature
  - Air speed
  - Humidity
- 
- The diagram uses red curly braces to group the factors. The first two factors, 'Metabolic rate (met)' and 'Clothing insulation (clo)', are grouped under the label 'Personal factors'. The remaining four factors, 'Air temperature', 'Radiant temperature', 'Air speed', and 'Humidity', are grouped under the label 'Environmental factors'.
- Personal factors
- Environmental factors

# Thermal comfort: 6 factors

- Metabolic rate (met)
  - Clothing insulation (clo)
  - Air temperature
  - Radiant temperature
  - Air speed
  - Humidity
- 
- The diagram shows a list of six factors for thermal comfort. The first two factors, 'Metabolic rate (met)' and 'Clothing insulation (clo)', are grouped by a red curly brace on the right and labeled 'Personal factors'. The remaining four factors, 'Air temperature', 'Radiant temperature', 'Air speed', and 'Humidity', are grouped by a yellow curly brace on the right and labeled 'Environmental factors'.
- Personal factors
- Environmental factors



# Personal factor #1: Metabolic rate

TABLE 5.2.1.2 Metabolic Rates	
Activity	Met Units
<b>Resting</b>	
Sleeping	0.7
Reclining	0.8
Seated, quiet	1.0
Standing, relaxed	1.2
<b>Walking (on level surface)</b>	
0.9 m/s, 3.2 km/h, 2.0 mph	2.0
1.2 m/s, 4.3 km/h, 2.7 mph	2.6
1.8 m/s, 6.8 km/h, 4.2 mph	3.8
<b>Office Activities</b>	
Reading, seated	1.0
Writing	1.0
Typing	1.1

# Personal factor #2: Clothing insulation

**TABLE 5.2.2.2B Garment Insulation ( $I_{clu}$ )**

Garment Description <sup>a</sup>	$I_{clu}(\text{clo})$	Garment Description <sup>a</sup>	$I_{clu}(\text{clo})$
Underwear		Dress and Skirts <sup>b</sup>	
Bra	0.01	Skirt (thin) mm	0.14
Panties	0.03	Skirt (thick)	0.23
Men's briefs	0.04	Sleeveless, scoop neck (thin)	0.23
T-shirt	0.08	Sleeveless, scoop neck (thick), i.e., jumper	0.27
Half-slip	0.14	Short-sleeve shirtdress (thin)	0.29
Long underwear bottoms	0.15	Long-sleeve shirtdress (thin)	0.33
Full slip	0.16	Long-sleeve shirtdress (thick)	0.47
Long underwear top	0.20	Sweaters	
Footwear		Sleeveless vest (thin)	0.13
Ankle-length athletic socks	0.02	Sleeveless vest (thick)	0.22
Panty hose/stockings	0.02	Long-sleeve (thin)	0.25
Sandals/thongs	0.02	Long-sleeve (thick)	0.36
Shoes	0.02	Suit Jackets and Vests <sup>c</sup>	
Slippers (quilted, pile lined)	0.03	Sleeveless vest (thin)	0.10

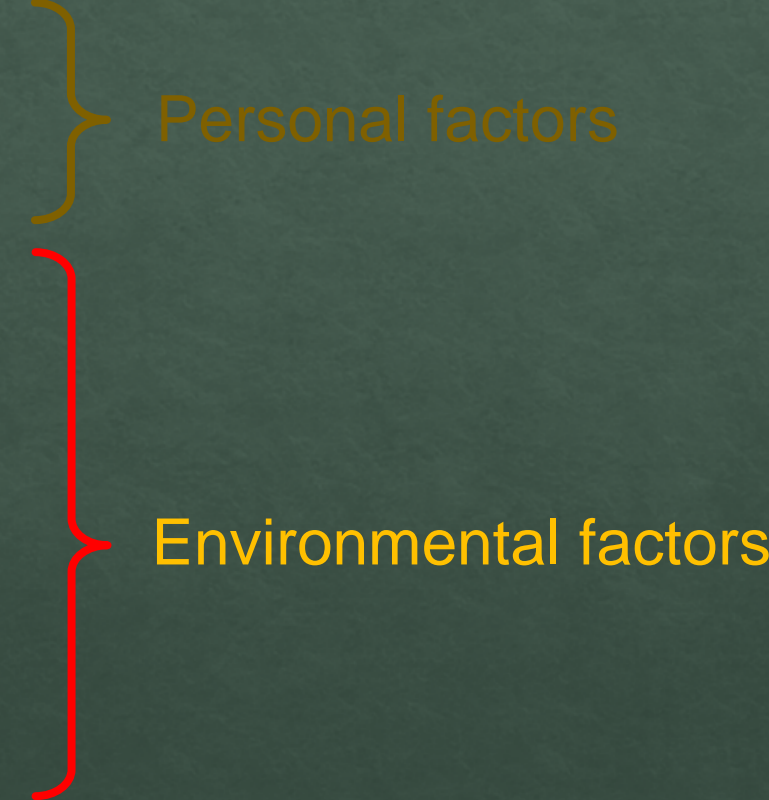


# Personal factor #2: Clothing insulation

**TABLE 5.2.2.2A Clothing Insulation ( $I_{cl}$ ) Values for Typical Ensembles**

Clothing Description	Garments Included*	$I_{cl}$ (clo)
Trousers	1) Trousers, short-sleeve shirt	0.57
	2) Trousers, long-sleeve shirt	0.61
	3) #2 plus suit jacket	0.96
	4) #2 plus suit jacket, vest, T-shirt	1.14
	5) #2 plus long-sleeve sweater, T-shirt	1.01
	6) #5 plus suit jacket, long underwear bottoms	1.30
Skirts/Dresses	7) Knee-length skirt, short-sleeve shirt (sandals)	0.54
	8) Knee-length skirt, long-sleeve shirt, full slip	0.67
	9) Knee-length skirt, long-sleeve shirt, half slip, long-sleeve sweater	1.10
	10) Knee-length skirt, long-sleeve shirt, half slip, suit jacket	1.04
	11) Ankle-length skirt, long-sleeve shirt, suit jacket	1.10
Shorts	12) Walking shorts, short-sleeve shirt	0.36
Overalls/Coveralls	13) Long-sleeve coveralls, T-shirt	0.72
	14) Overalls, long-sleeve shirt, T-shirt	0.89

# Thermal comfort: 6 factors

- Metabolic rate (met)
  - Clothing insulation (clo)
  - Air temperature
  - Radiant temperature
  - Air speed
  - Humidity
- 
- The diagram uses two large curly braces on the right side to group the factors. A brown brace groups the first two factors (Metabolic rate and Clothing insulation) under the label 'Personal factors'. A red brace groups the remaining four factors (Air temperature, Radiant temperature, Air speed, and Humidity) under the label 'Environmental factors'.
- Personal factors
- Environmental factors



# Envir. factor #1: Air temperature

- The most obvious
- Most thermostats do this (and only this)

# Envir. factor #1: Air temperature

- The most obvious
- Most thermostats do this (and only this)
- “Universal compensator” for the other 5 variables (personal and environmental)



## Envir. factor #2: Radiant temperature

- The “hot” and “cold” of surrounding surfaces can make you uncomfortable!
- This is why it’s possible to feel cold in a 72F room when seated near a cool window or wall

## Envir. factor #2: Radiant temperature

### Implications:

- Poor windows and cold walls have real comfort impacts
- It's also why radiant or insulated floors can help even if air temp isn't necessarily high

Mean Radiant Temperature (MRT) is a measurement that takes surrounding surfaces into account (weighted average surrounding a single point)

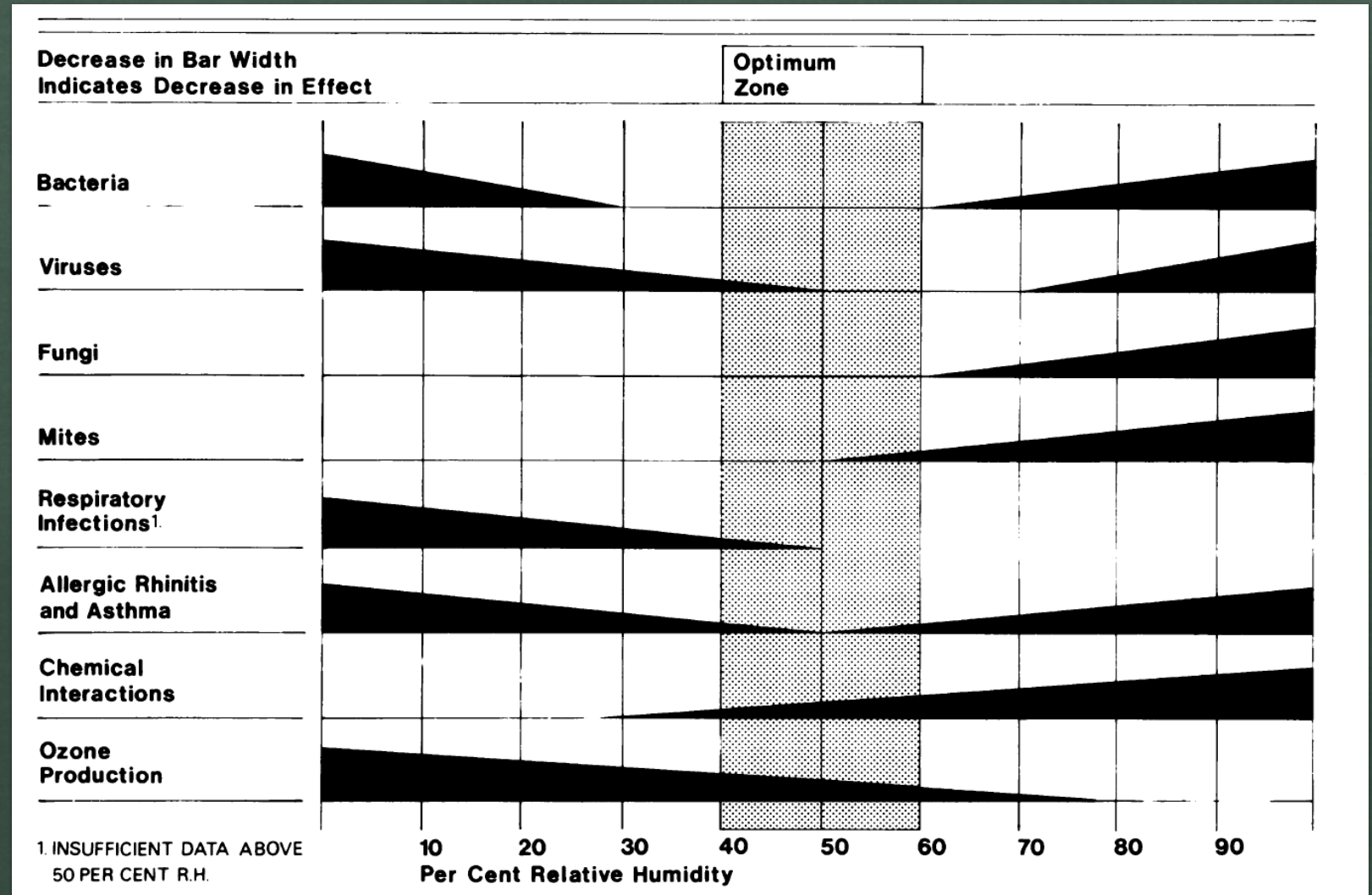




# Envir. factor #3: Humidity

Sterling chart –  
Useful but take  
with a grain of salt  
A decent target:  
40%

Ref: [Humidity, Health, and the Sterling chart](#), Energy Vanguard



# Envir. factor #3: Humidity

And more...

why control rh between 35% & 55% (+/-5%)

to control hydrolysis (VOC emissions)

to control microbial (virus, bacteria, molds, mites, some insects)

to enable comfort in mucous membranes (respiratory, eyes (skin))

to support hydration (affects cognition and wound healing)

to enable positive perceptions of thermal comfort

to enable positive perceptions of indoor air quality

to enable positive perceptions of indoor odour quality

to maintain dimensional stability in hygroscopic materials (woods)

to prevent condensation on hydrophobic materials (glass)

to prevent condensation in hydrophilic materials (drywall)

to preserve moisture sensitive artifacts / collectibles / musical instruments



## Envir. factor #4: Air motion

- Good for delivery and mixing
- However, it *can* be bad when
  - In winter, cool moving air that's more than 3F below room temperature\*
  - Velocities greater than 30 fpm\*

\*Per ASHRAE 55

# Other definitions

## Dry bulb temperature

- “Air temperature” (typ.) – shielded from radiation and moisture

## Wet bulb temperature

- Temperature a parcel of air would have if it were cooled to saturation (100% RH) by evaporation of water into it, with latent heat supplied by the parcel
- At 100% RH, wet bulb temp = dry bulb temp
- Sling psychrometer





# Other definitions

Operative temperature

- $T_{op} = \frac{1}{2} \text{ dry bulb} + \frac{1}{2} \text{ MRT}$

We'll see this a lot...

# ASHRAE 55 comfort framework



# PMV and PPD

PMV = Predicted Mean Vote

- ASHRAE scale (subjective)
- “Comfort zone” defined as  $-0.5$  PMV to  $+0.5$  PMV



+3	Hot
+2	Warm
+1	Slightly warm
0	Neutral
-1	Slightly cool
-2	Cool
-3	Cold

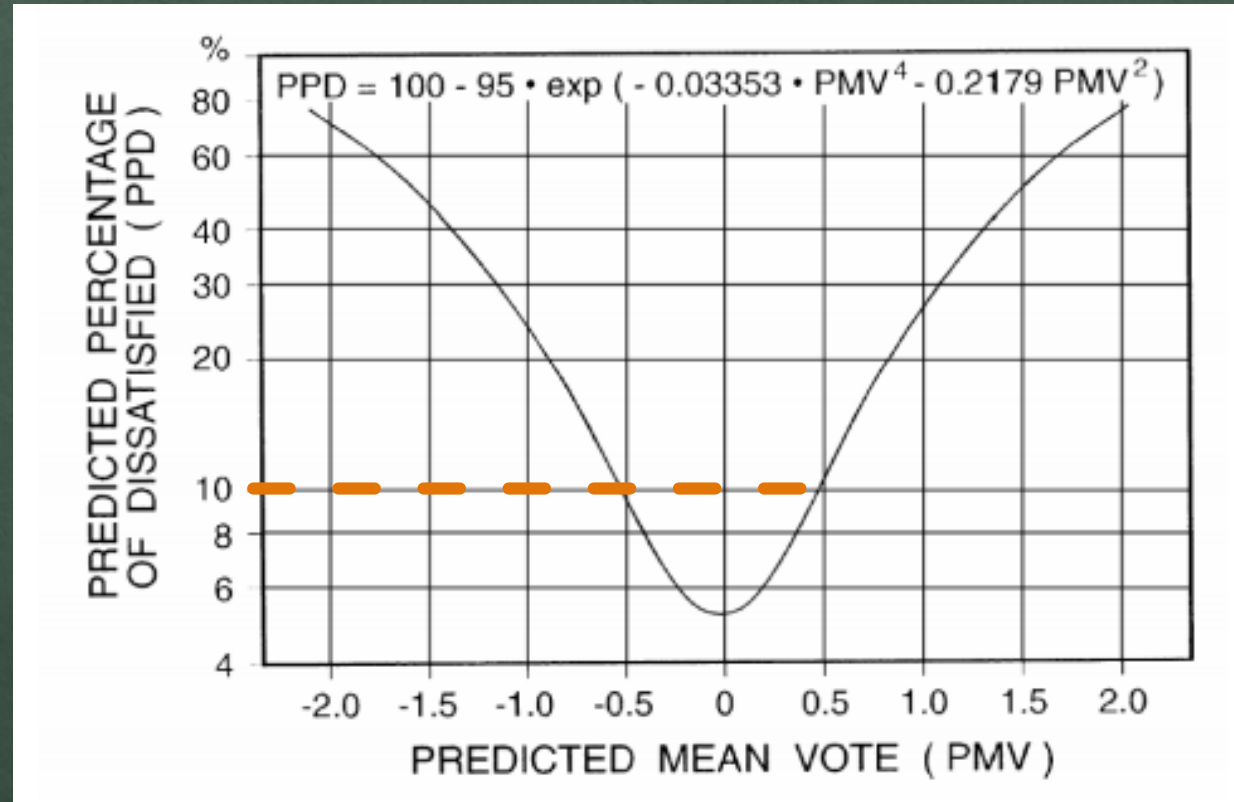
PPD = Predicted Percentage Dissatisfied

- People voting  $|2|$  or greater



Empirical model  $-0.5$  PMV to  $+0.5$  PMV predicts 90% of population satisfied, or 10% PPD

# PMV and PPD





# Local discomfort

We want to avoid:

1. Radiant temperature asymmetry

- Ceiling  $\geq 9\text{F}$  warmer than floor
- Ceiling  $\geq 14\text{F}$  cooler than floor
- Wall  $\geq 41\text{F}$  warmer than air
- Wall  $\geq 18\text{F}$  cooler than air

2. Draft

- If  $\text{clo} < 0.7$ ,  $\text{met} < 1.3$ , and  $T_{\text{op}} < 72.5\text{F}$ , keep  $< 30\text{ fpm}$
- Upper limit is 240 fpm with right conditions



# Local discomfort

We want to avoid:

3. Vertical air temperature difference
  - Head and ankle difference  $< 5.4^{\circ}\text{F}$
4. Floor surface temperature too hot or too cold
  - Comfort range *while wearing shoes* is  $66.2 - 84.2^{\circ}\text{F}$
5. Temperature variation with time
6. Cyclic variations, drifts, and ramps
  - Operative temperature should vary
    - Within  $2^{\circ}\text{F}$  if cycling within a 15-minute period
    - Within  $4^{\circ}\text{F}$  over 1-hour period



# Huh?

Bottom line =

- This is about occupants and it's subjective
- The target is to keep 80% of people happy
- There *are* some “hard” guidelines
- **...and there are real implications with significant overlap with energy efficiency**



Thank you,  
Ole Fanger.

Now...let's see  
how to use this.



# ASHRAE 55: Application

## Primary tools

1. Graphical method (chart)
2. Software method  
(CBE Thermal Comfort tool,  
<https://comfort.cbe.berkeley.edu/>)



## Example: Brian's house in winter (graphical method)

- Thermostat 66F
- RH 30%
- Clo = 0.90 (typ. winter outfit)
- Met = 1.0 (seated, reading)

$$\begin{aligned}T_{op} &= \frac{1}{2} \text{ dry bulb} + \frac{1}{2} \text{ MRT} \\&= \frac{1}{2} 66\text{F} + \frac{1}{2} ?? \text{ (don't know...yet)} \\&= \text{For now, assume it's about } 66\text{F}\end{aligned}$$

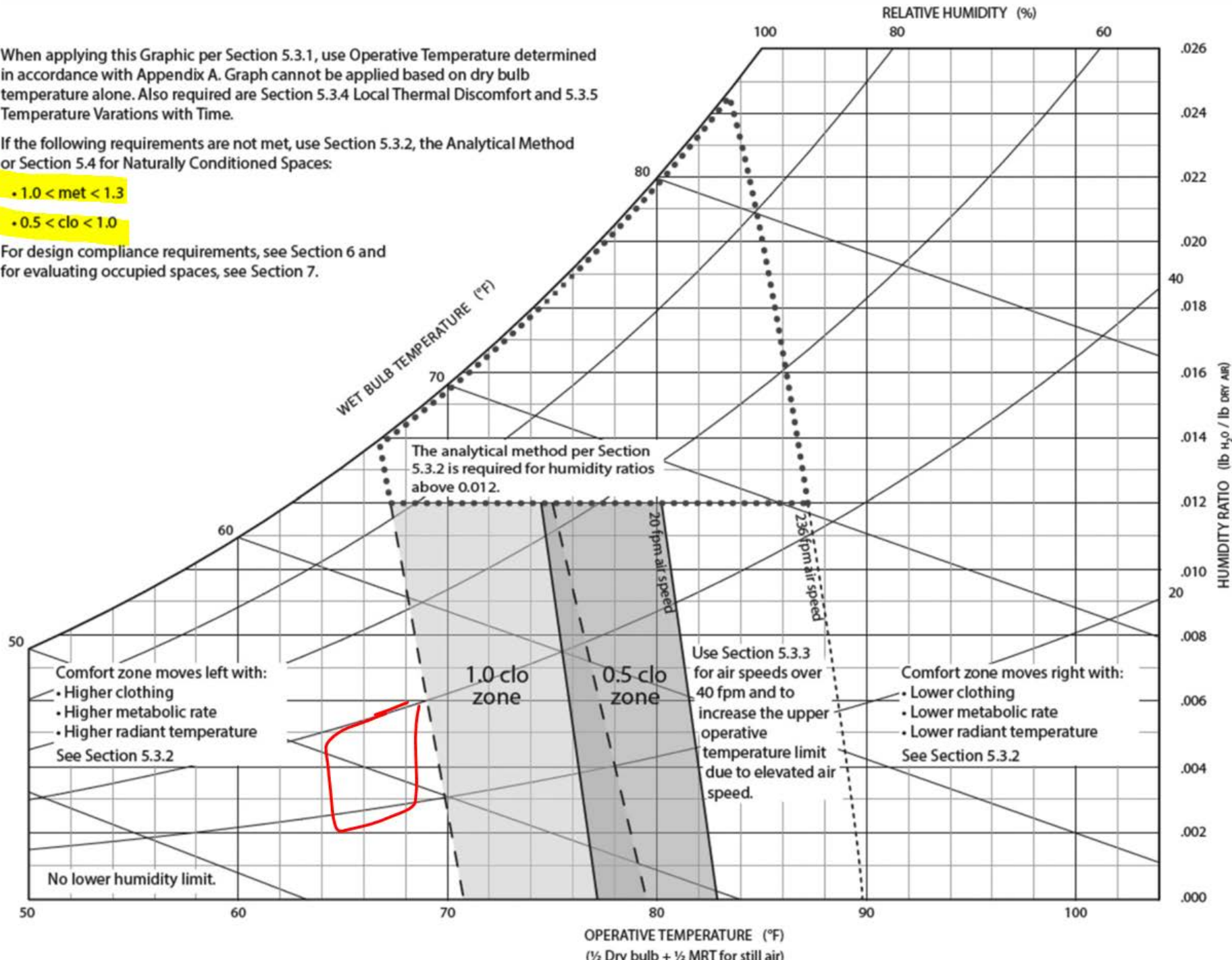
When applying this Graphic per Section 5.3.1, use Operative Temperature determined in accordance with Appendix A. Graph cannot be applied based on dry bulb temperature alone. Also required are Section 5.3.4 Local Thermal Discomfort and 5.3.5 Temperature Variations with Time.

If the following requirements are not met, use Section 5.3.2, the Analytical Method or Section 5.4 for Naturally Conditioned Spaces:

•  $1.0 < \text{met} < 1.3$

•  $0.5 < \text{clo} < 1.0$

For design compliance requirements, see Section 6 and for evaluating occupied spaces, see Section 7.





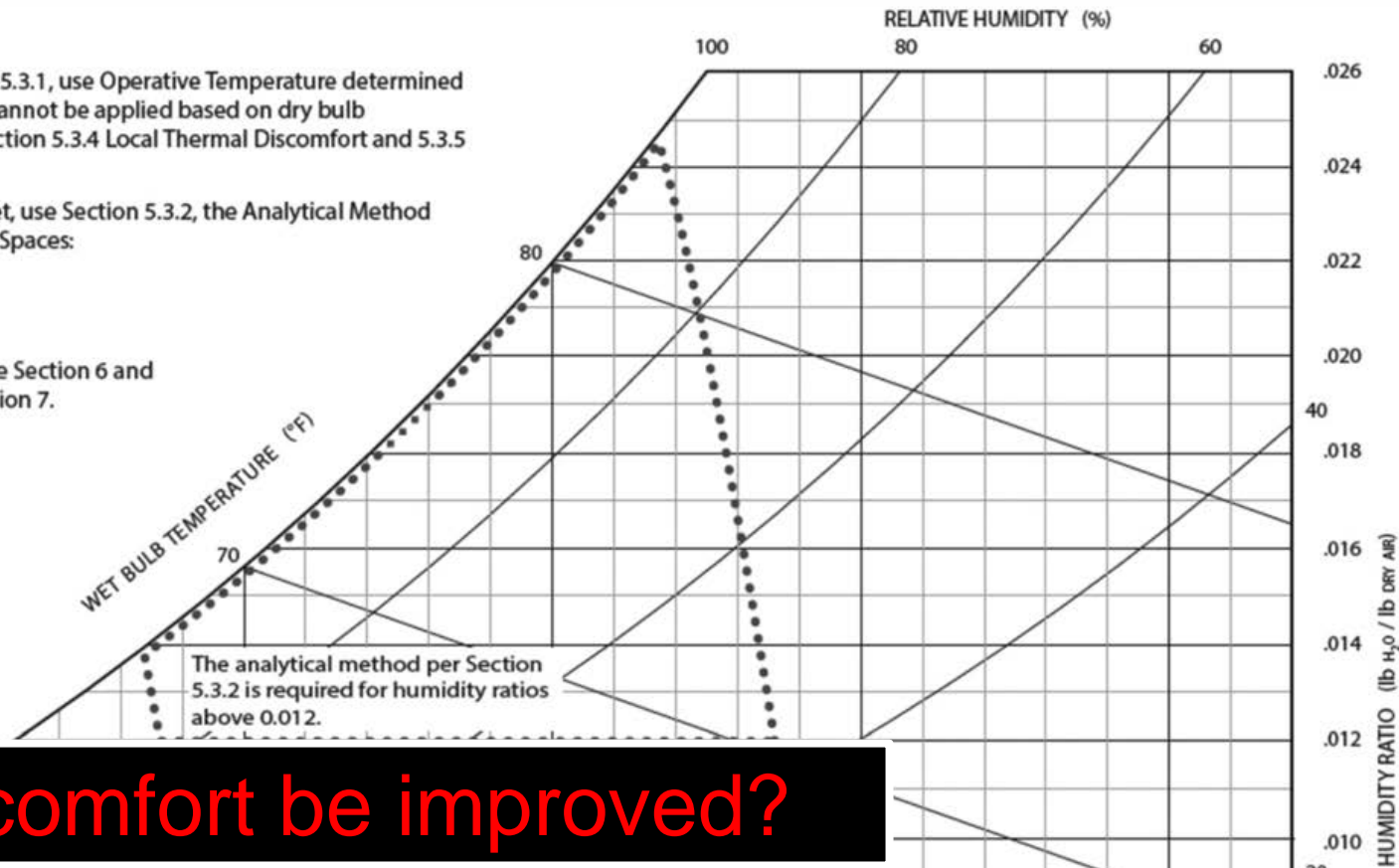
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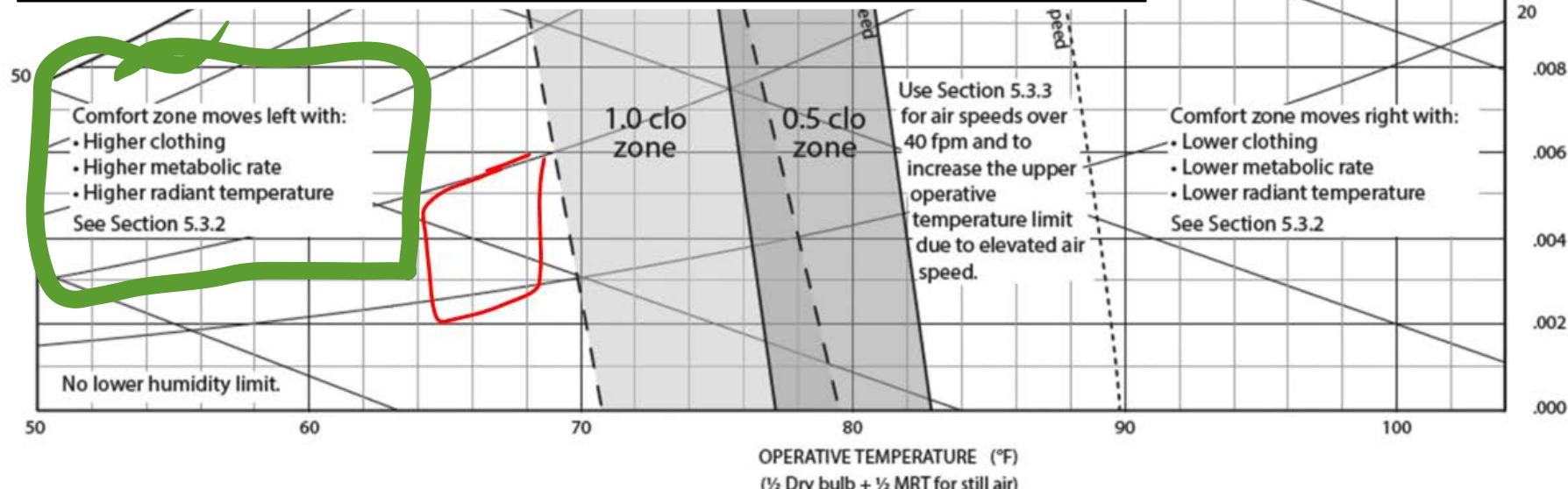
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•  $0.5 < \text{clo} < 1.0$

For design compliance requirements, see Section 6 and for evaluating occupied spaces, see Section 7.



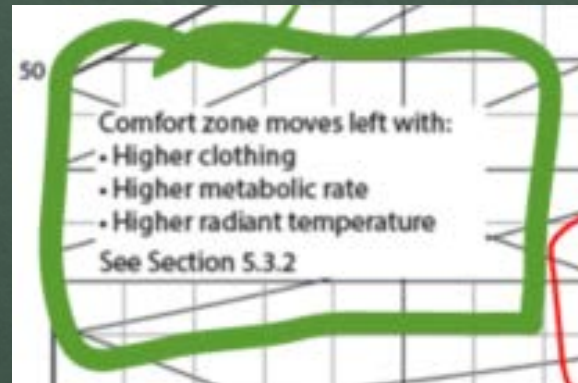
# How can comfort be improved?



## Example: Brian's house in winter (graphical method)

Tools for improving comfort when cold:

- Higher clothing
- Higher metabolic rate
- Higher MRT



- But if we tweak these too much, we're easily pushing off the “valid” scale of the chart – so we need another tool



# Example: Brian's house in winter (software method)

- Thermostat 66F
- RH 30%
- Clo = 0.90 (typ. winter)
- Met = 1.0 (seated, reading)

$$\begin{aligned}T_{op} &= \frac{1}{2} \text{ dry bulb} + \frac{1}{2} \text{ MRT} \\&= \frac{1}{2} 66\text{F} + \frac{1}{2} ?? \text{ (don't know...yet)} \\&= \text{For now, assume it's about 66F}\end{aligned}$$

The screenshot shows the 'CBE Thermal Comfort Tool' interface. The header includes the CBE logo and the text 'CBE Thermal Comfort Tool', 'ASHRAE-55', 'EN-16798', 'Compare', and 'Ranges'. The 'Inputs' section contains the following fields and controls:

- Select method:** A dropdown menu set to 'PMV method'.
- Operative temperature:** A numeric input field set to '66' with a unit dropdown set to '°F'.
- Air speed:** A numeric input field set to '19.7' with a unit dropdown set to 'fpm'.
- Relative humidity:** A numeric input field set to '30' with a unit dropdown set to '%'. To the right is a dropdown menu labeled 'Relative humidity'.
- Metabolic rate:** A numeric input field set to '1' with a unit dropdown set to 'met'. To the right is a dropdown menu labeled 'Reading, seated: 1.0'.
- Clothing level:** A numeric input field set to '1' with a unit dropdown set to 'clo'. To the right is a dropdown menu labeled 'Typical winter indoor'.

Below the input fields are several buttons: 'Create custom ensemble', 'Dynamic predictive clothing', 'Solar gain on occupants', 'Set pressure' (with a red circle around the 'SI/IP' dropdown), 'Local discomfort', and 'Globe temp'. The bottom right corner of the interface shows the text 'NOT'.

# Example: Calculating MRT

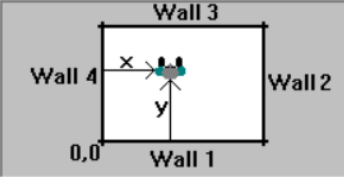
Remember, this is simply combining all the temperatures around you into a single value

1. Obsolete ASHRAE software →
2. Somewhat tedious, metric-only free online software  
<http://centerforthebuiltenvironment.github.io/mrt/>
3. Back of the envelope

MRT Calculator

Room dimensions  
Room width (x)  ft  
Room length (y)  ft  
Room height  ft

Occupant  
x  ft  
y  ft  
Facing   
Azimuth   
☒ Seated ☐ Standing



Glass/panel data

	Temperature °F	Emis	Width ft	Height ft	Centered	Sill ft	L. Jamb* ft	Window view factor
Wall 1	<input type="text" value="66.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.059"/>
	<input type="checkbox"/> window/panel							
Wall 2	<input type="text" value="66.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.075"/>
	<input checked="" type="checkbox"/> window/panel							
Wall 3	<input type="text" value="66.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.201"/>
	<input checked="" type="checkbox"/> window/panel							
Wall 4	<input type="text" value="66.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.060"/>
	<input checked="" type="checkbox"/> window/panel							
Ceiling	<input type="text" value="66.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.149"/>
	<input type="checkbox"/> window/panel							
Floor	<input type="text" value="60.0"/>	<input type="text" value="0.90"/>						<input type="text" value="0.309"/>
	<input type="checkbox"/> panel							
View factor total	<input type="text" value="1.0"/>							
MRT	<input type="text" value="62.7"/> °F							

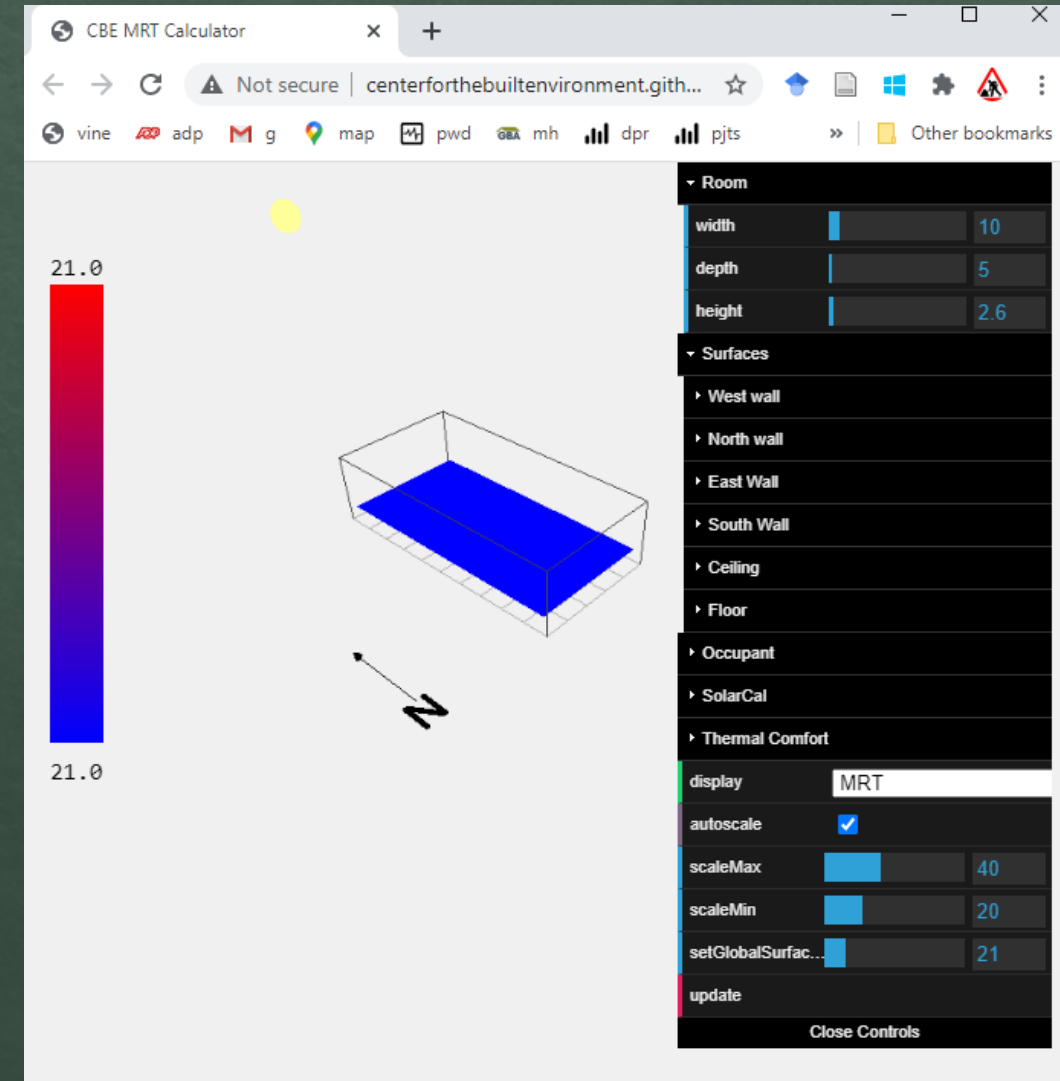
\*distance from left edge of wall to left jamb when viewed from inside the room  
View factors are calculated based on Fanger, P.D., "Thermal Comfort", McGraw-Hill, 1972



# Example: Calculating MRT

Remember, this is simply combining all the temperatures around you into a single value

1. Obsolete ASHRAE software
2. Somewhat tedious, metric-only free online software → <http://centerforthebuiltenvironment.github.io/mrt/>
3. Back of the envelope



# Example: Calculating MRT

Remember, this is simply combining all the temperatures around you into a single value

1. Obsolete ASHRAE software
2. Somewhat tedious, metric-only free online software  
<http://centerforthebuiltenvironment.github.io/mrt/>
3. Back of the envelope →

Standing in middle of a huge room:

- 30% exposure to 60F floor
- 30% exposure to 66F ceiling
- 35% exposure to 66F walls
- 5% exposure to 55F windows

$$\text{MRT} = 0.30 \cdot 60\text{F} + 0.30 \cdot 66\text{F} + 0.35 \cdot 66\text{F} + 0.05 \cdot 55\text{F} = 63.65\text{F}$$

Sitting 2' from a window in a living room:

- 30% exposure to 60F floor
- 15% exposure to 66F ceiling
- 40% exposure to 66F walls
- 15% exposure to 55F windows

$$\text{MRT} = 0.30 \cdot 60\text{F} + 0.15 \cdot 66\text{F} + 0.40 \cdot 66\text{F} + 0.15 \cdot 55\text{F} = 62.55\text{F}$$



# Example: Brian's house in winter (software method)

- Thermostat 66F
- RH 30%
- Clo = 0.90 (typ. winter)
- Met = 1.0 (seated, reading)

$$\begin{aligned}T_{op} &= \frac{1}{2} \text{ dry bulb} + \frac{1}{2} \text{ MRT} \\&= \frac{1}{2} 66\text{F} + \frac{1}{2} 62.7\text{F} \\&= 64.35\text{F}\end{aligned}$$

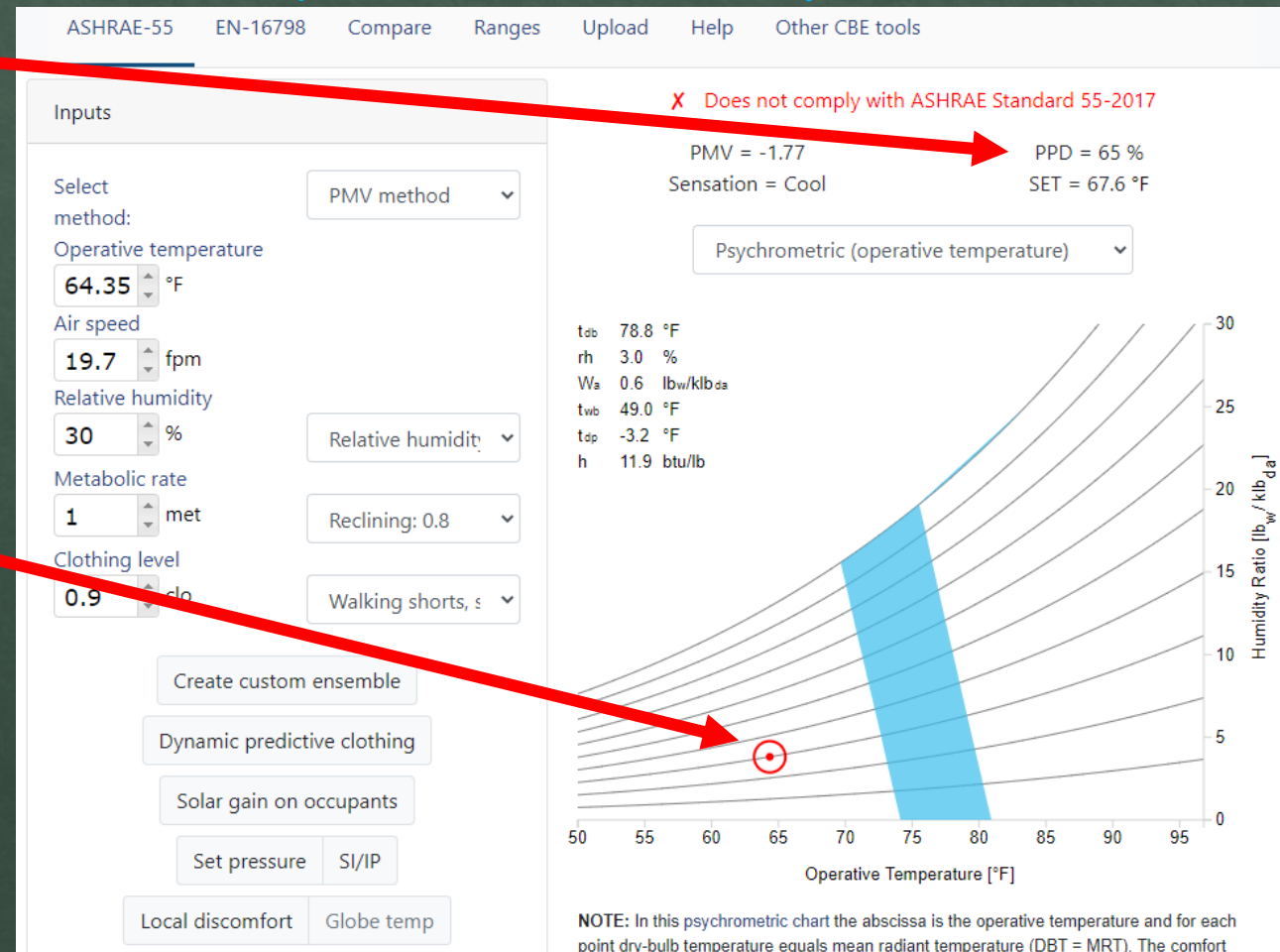
Enter into: <https://comfort.cbe.berkeley.edu/>



# Let's unpack this...

- 65% of people are predicted to find it uncomfortable
- Misses ASHRAE 55 by a long shot (you want to be in the blue zone)

Enter into: <https://comfort.cbe.berkeley.edu/>





# How do you fix it?

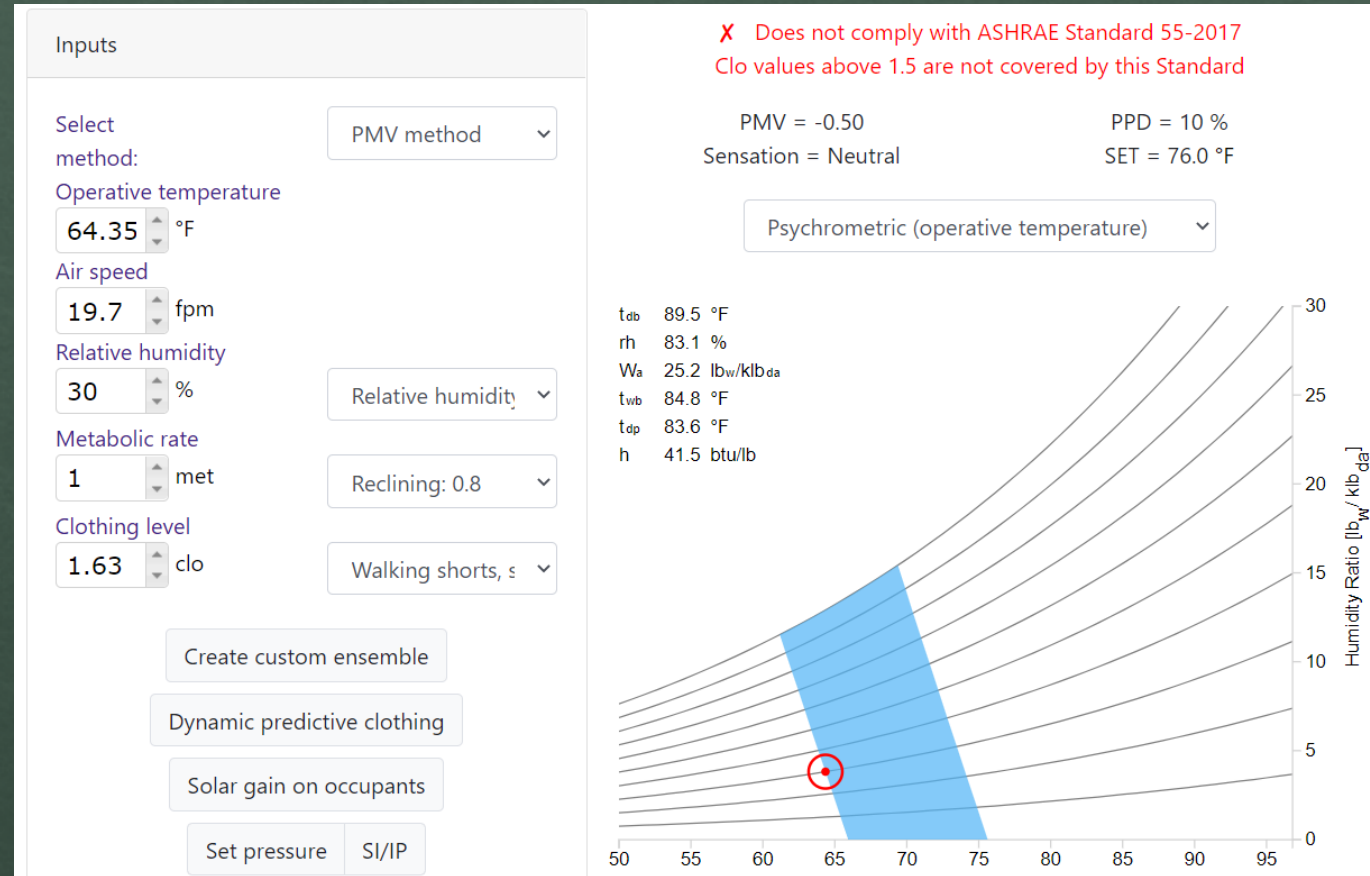
## Tools:

- More clothing
- Higher metabolic rate
- Higher MRT (increase surface temperatures)
- Cranking the thermostat (but first let's not)

## ...via more clothes

All things equal, I need to reach  $Clo = 1.63$  to get to “neutral” comfort

For example, adding 2 thick sweaters (adds 0.36 each to  $Clo$ )

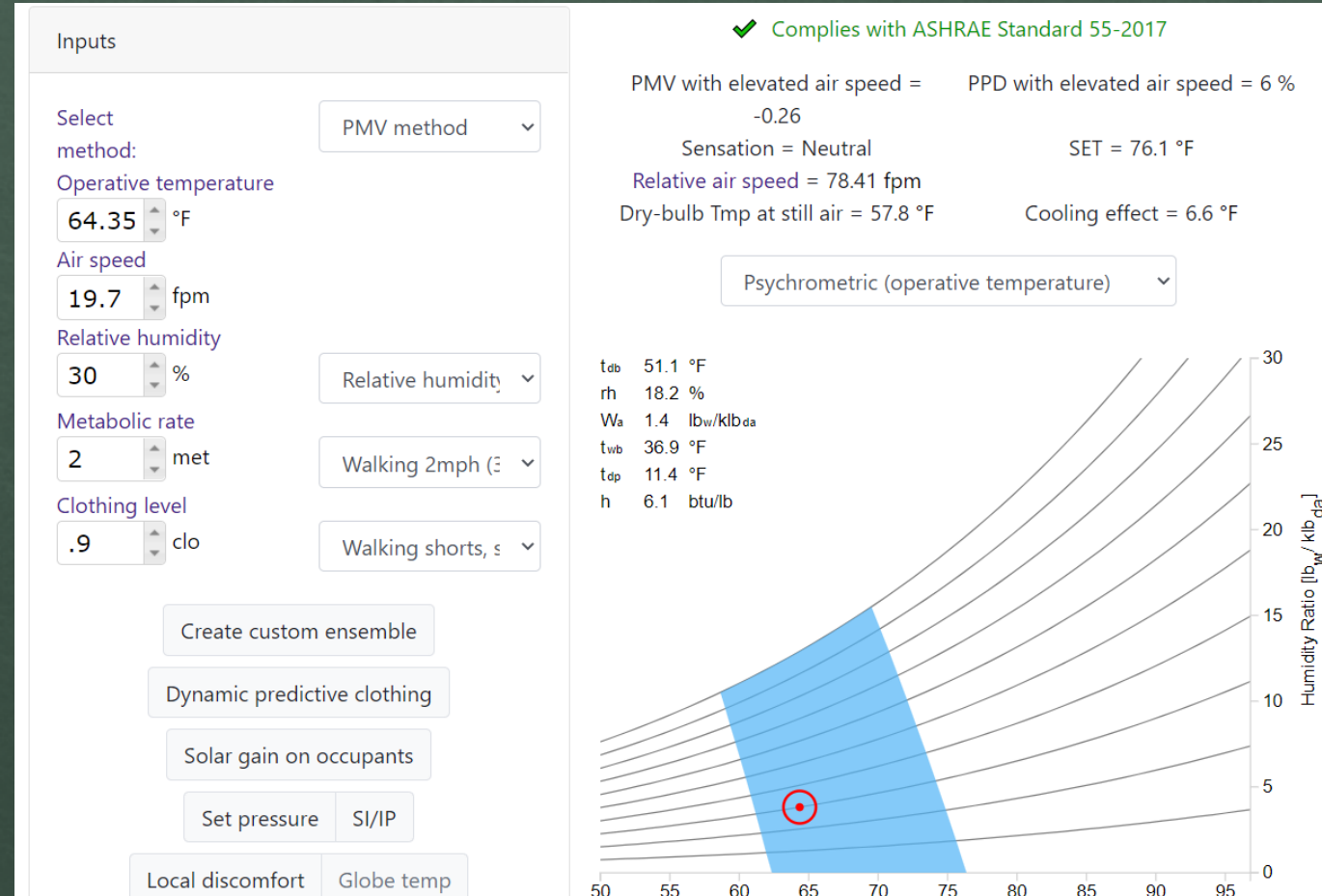




## ...via higher metabolic rate

Jump to about 2.0 met value

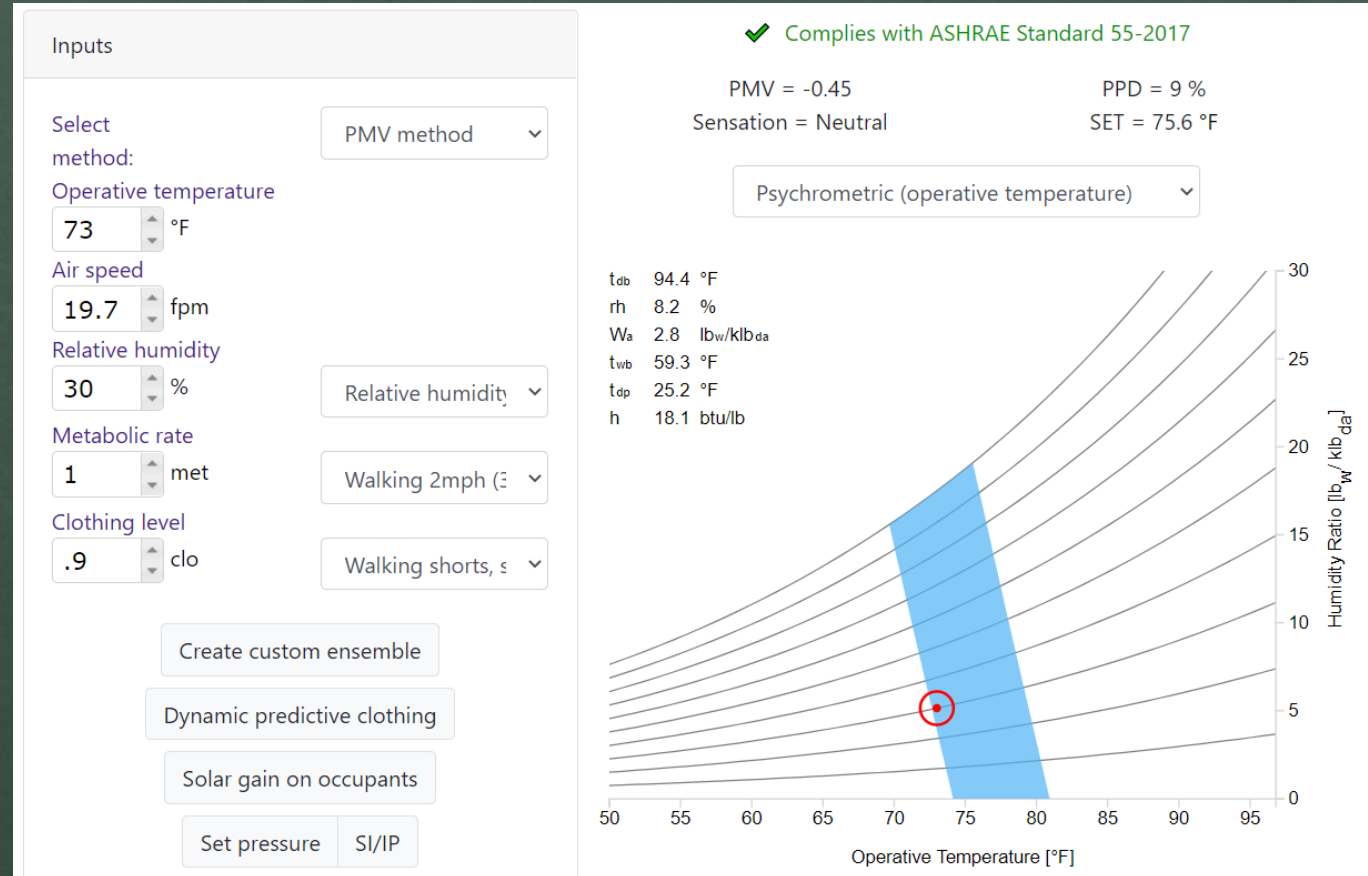
For example, walking at 2 mph (an activity like cooking gets you close but not quite)



# ...via higher MRT

Increase  $T_{op}$  from 64.35F to 73F

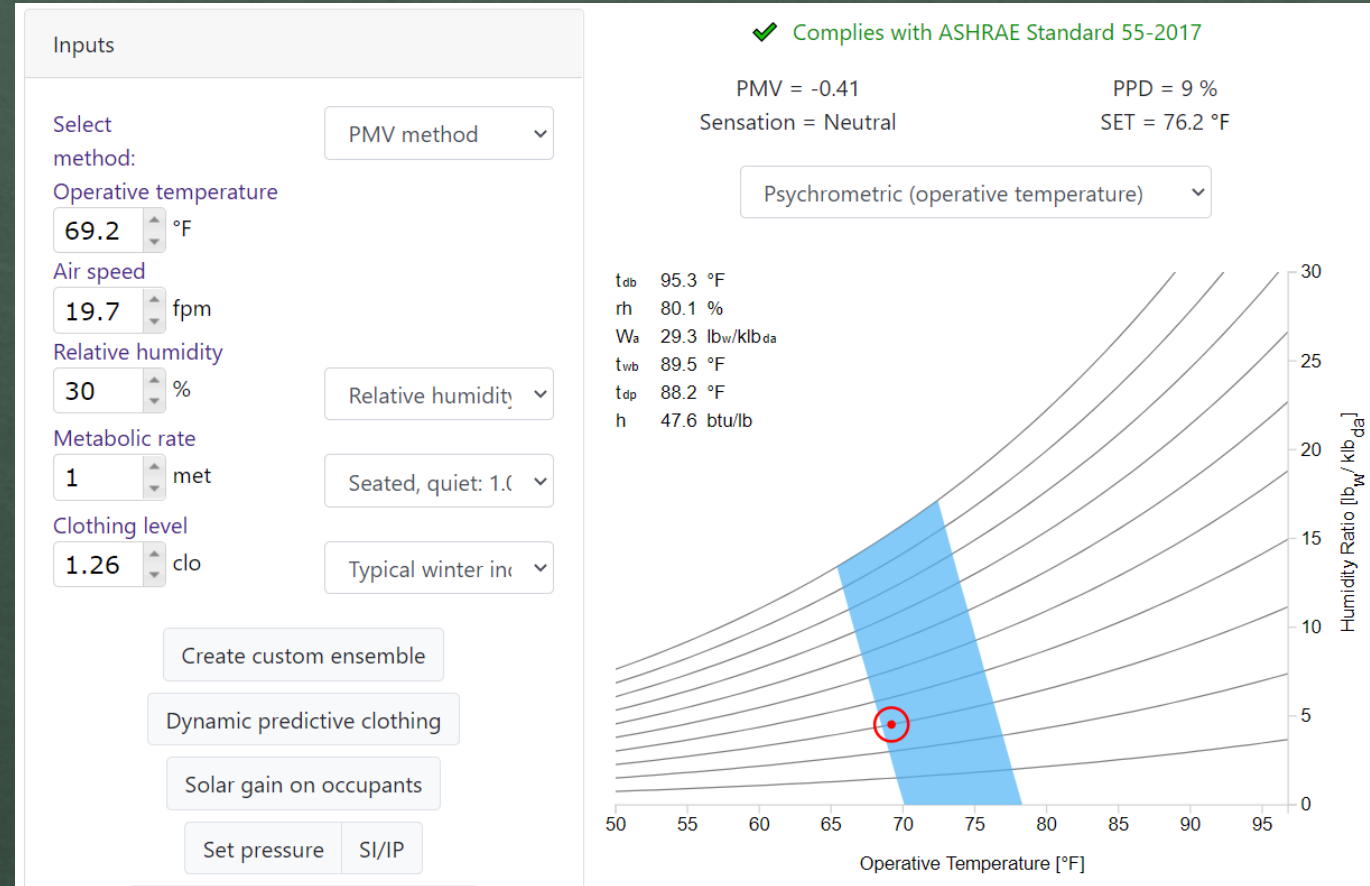
- 90F radiant floor gets you to about MRT 72.5,  $T_{op} = 69.25 \rightarrow$  not enough
- 90F radiant floor plus room temp of 73.5F does it, however
- But really?





# More realistic

- Thermostat 68F
  - With 85F radiant floor
  - MRT ~ 70.3F,  $T_{op}$  ~ 69.2F
- RH 30%
- Clo = 1.26 (winter indoors wear plus warm sweater)
- Met = 1.0 (seated, reading)



# Remember to also consider

## Radiant temperature asymmetry

- Ceiling  $\geq 9\text{F}$  warmer than floor
- Ceiling  $\geq 14\text{F}$  cooler than floor
- Wall  $\geq 41\text{F}$  warmer than air
- Wall  $\geq 18\text{F}$  cooler than air

## Draft

- If  $\text{clo} < 0.7$ ,  $\text{met} < 1.3$ , and  $T_{\text{op}} < 72.5\text{F}$ , keep  $< 30$  fpm
- Upper limit is 240 fpm with right conditions

## Vertical air temperature difference

- Head and ankle difference  $< 5.4\text{F}$

## Floor surface temperature

- Comfort range *while wearing shoes* is  $66.2\text{F} - 84.2\text{F}$

## Cyclic variations, drifts, and ramps

- Operative temperature varies
  - Within  $2\text{F}$  if cycling within a 15-minute period
  - Within  $4\text{F}$  over 1-hour period



# Tools beyond ASHRAE

# Useful tools

1. Graphic comfort zone method (chart)
2. CBE Thermal Comfort tool,  
<https://comfort.cbe.berkeley.edu/>
3. THERM or other simulation software
4. Payette glazing tool
5. Cardinal tool
6. Free stuff! (for later)

} Covered  
already



# THERM

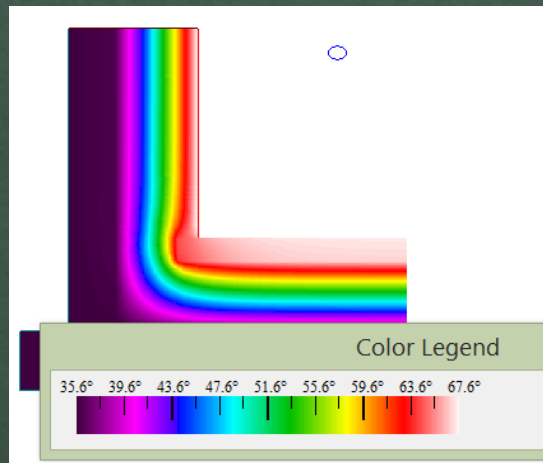
2D heat transfer modeling software

- Lawrence Berkeley National Laboratory
- Free, <https://windows.lbl.gov/software/therm>

Example: Residential slab insulation model...

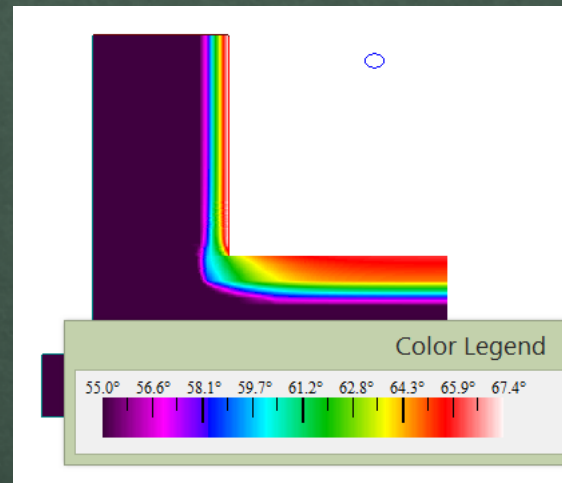
# R-30

## Slab 67F



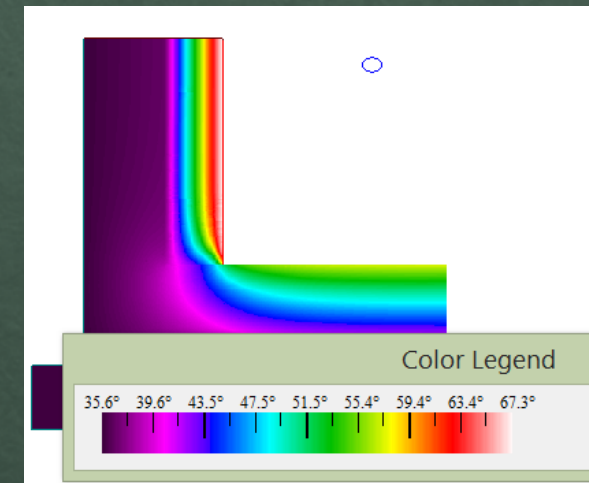
# R-10

## Slab 63-65F



# R-1

## Slab 52-57F





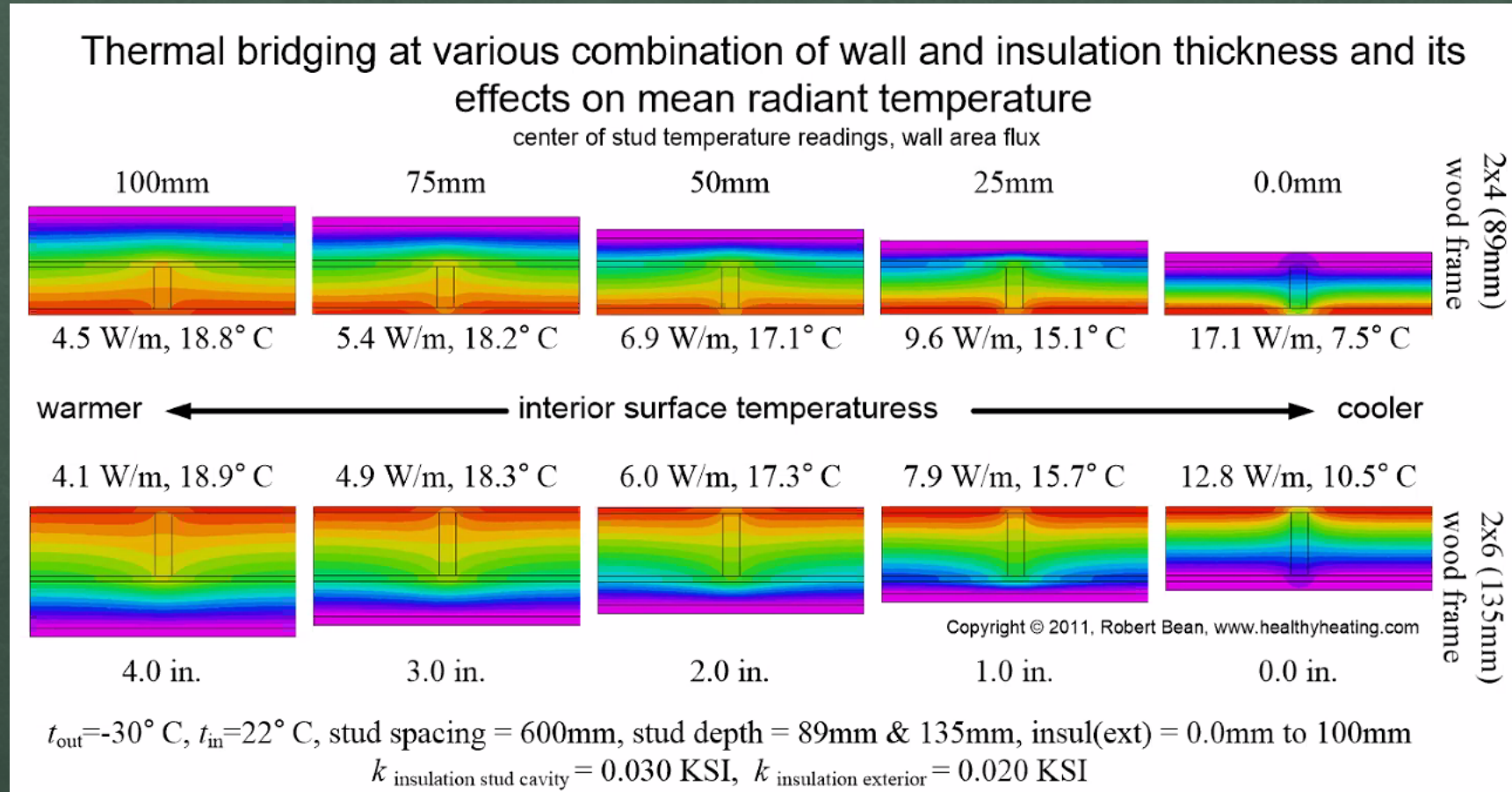
# Recall...

- Local discomfort due to radiant temperature asymmetry when...
  - Ceiling  $\geq 9\text{F}$  warmer than floor
- Floor surface temperature
  - Comfort range while wearing shoes is  $66.2\text{F} - 84.2\text{F}$

Yet many codes (e.g. Vermont) do not require under-slab insulation for unheated slabs

- Result: Slab temp in the 50s  $\rightarrow$  discomfort
- Think about MRT/comfort (and potential for summer condensation and associated problems)

# Thermal bridging / continuous insulation effects on MRT





### Thermal bridging at various combination of wall and insulation thickness and its effects on mean radiant temperature

center of stud temperature readings, wall area flux

Insulation Thickness (mm)	Insulation Thickness (in.)	Heat Flux (W/m)	Center of Stud Temperature (°C)	Center of Stud Temperature (°F)
100mm	4.0 in.	4.5 W/m	18.8°C	66°F
75mm	3.0 in.	5.4 W/m	18.2°C	65°F
50mm	2.0 in.	6.9 W/m	17.1°C	63°F
25mm	1.0 in.	9.6 W/m	15.1°C	60°F
0.0mm	0.0 in.	17.1 W/m	7.5°C	51°F

warmer ← interior surface temperatures → cooler

wood frame 2x4 (89mm) | wood frame 2x6 (135mm)

Copyright © 2011, Robert Bean, [www.healthyheating.com](http://www.healthyheating.com)

$t_{out} = -30^{\circ}\text{C}$ ,  $t_{in} = 22^{\circ}\text{C}$ , stud spacing = 600mm, stud depth = 89mm & 135mm, insul(ext) = 0.0mm to 100mm  
 $k_{\text{insulation stud cavity}} = 0.030 \text{ KSI}$ ,  $k_{\text{insulation exterior}} = 0.020 \text{ KSI}$

# Payette glazing tool

<https://www.payette.com/glazing-and-winter-comfort-tool/>

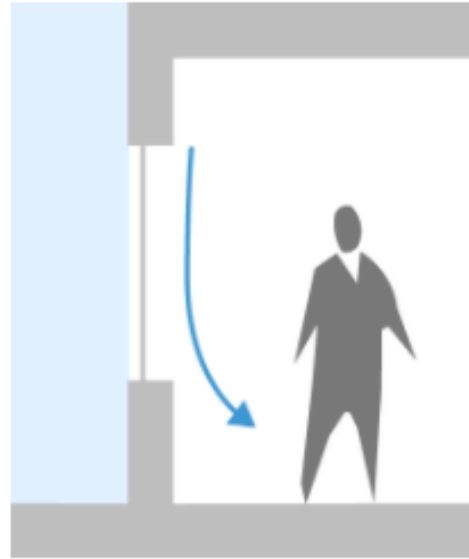
The software models:

- Radiant discomfort
- Draft discomfort\*

Payette note: Perimeter heat warms inner glass, which has the effect of *raising* effective u-factor (wrong way!)

\* ASHRAE tools do not account for this

## Understanding Discomfort

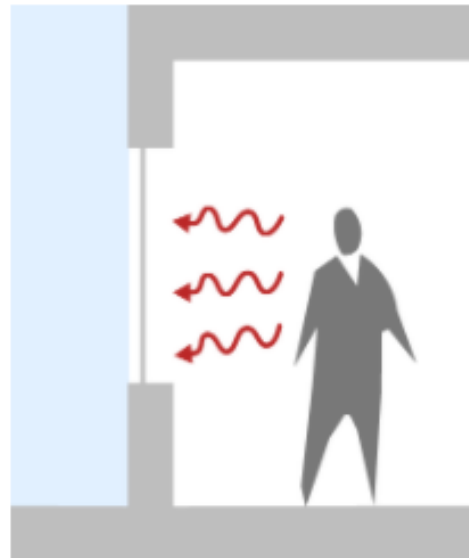


### DOWNDRAFT DISCOMFORT

Cold convective air currents, formed by warm room air hitting the cold window surface, create discomfort at the occupant's feet and ankles. The strength of these currents depends on the height of the window pane, as well as the interior temperature of the glass.

To minimize downdraft discomfort, try:

- decreasing the window height
- decreasing the window U-Value
- using a glazing assembly without a room-side low-e coating



### RADIANT DISCOMFORT

Cold interior glass surfaces affect the mean radiant temperature of occupants, and in turn make them feel cold. This discomfort depends on how much the occupant “sees” of the glass, how cold the interior glass surface is and the emissivity of the glass. If the glass has a room-side low-e coating the radiant discomfort will be greatly reduced.

To minimize radiant discomfort, try:

- decreasing the total amount of glazing
- decreasing the window U-Value
- increasing the sill height
- using a glazing assembly with a room-side low-e coating



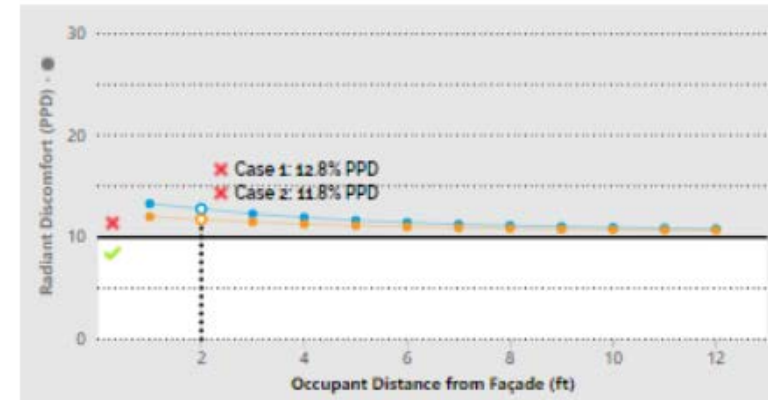
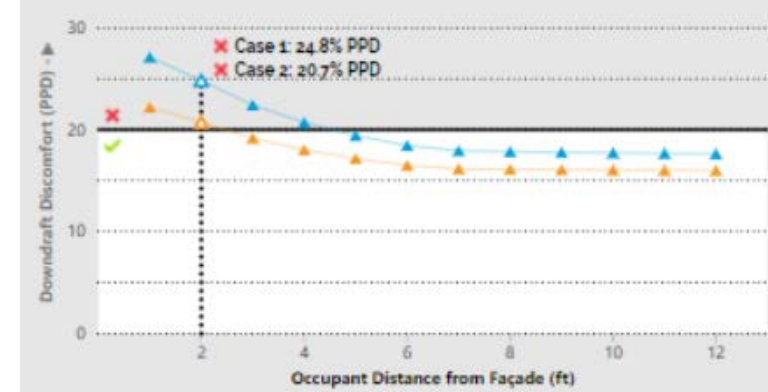
# Payette glazing tool

<https://www.payette.com/glazing-and-winter-comfort-tool/>

## Sample scenario

- Blue = code walls and windows
- Yellow = high performance walls, triple-glazed windows

Note: Does not take solar radiation into account – arguably, this can warm an occupant in winter (if orientation and time of day are aligned), but it's (probably) impossible to design a building to count on this (barring 360-degree windows or building rotation, and only occupied during daytime)



## GRAPH TYPE

☒ Split  
☐ Combined

## OUTDOOR DESIGN CONDITION

Outdoor Temperature (°F)

## FAÇADE GEOMETRY

Ceiling Height (ft)

Room Length (ft)

Window Height From Sill (ft)

Sill Height (ft)

Set Glazing Amount By

☒ Window Width (ft)

☐ Window-to-Wall Ratio (%)

Window Separation (ft)

## UNITS

☒ IP ☐ SI

## FAÇADE PERFORMANCE

Window U-Value (Btu/ft<sup>2</sup>·hr·°F)

U-Value that meets the target PPD

Is there a risk of condensation?

## INDOOR CONDITIONS

Indoor Temperature (°F)

Relative Humidity (%)

## SHARE

## ADVANCED OPTIONS

Room-side Low-E Coating ☐ ☐

Emissivity

Wall R-Value (ft<sup>2</sup>·hr·°F/Btu)

Air Speed (fpm)

Clothing (clo)

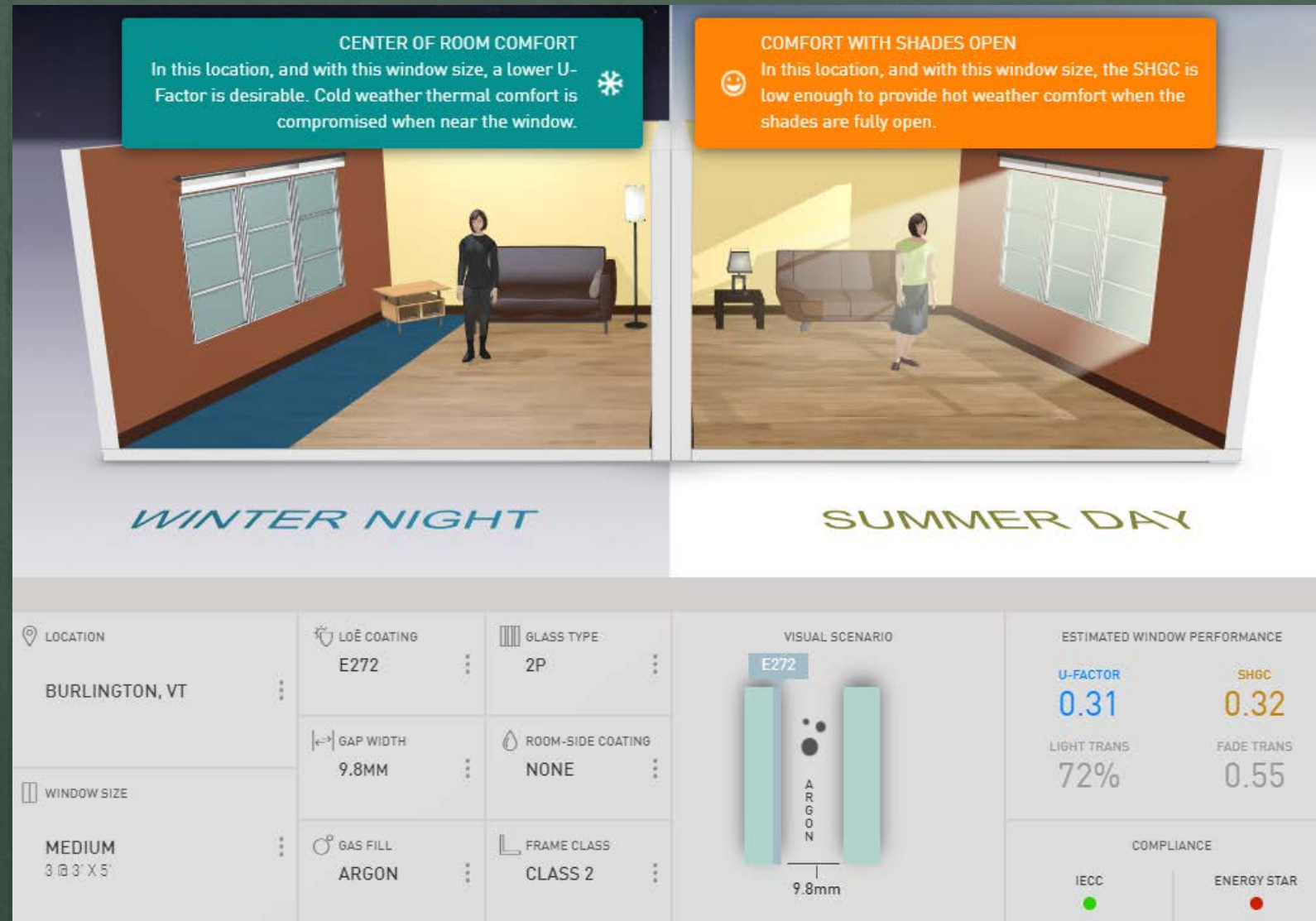
Metabolic Rate (met)

# Cardinal glazing tool

<https://www.cardinalcorp.com/technology/applications/comfort-calculator/>

The software only accounts for glass (and it goes into the weeds)

In this scenario, a double pane window results in discomfort anywhere within 4-5' of a window on a winter night



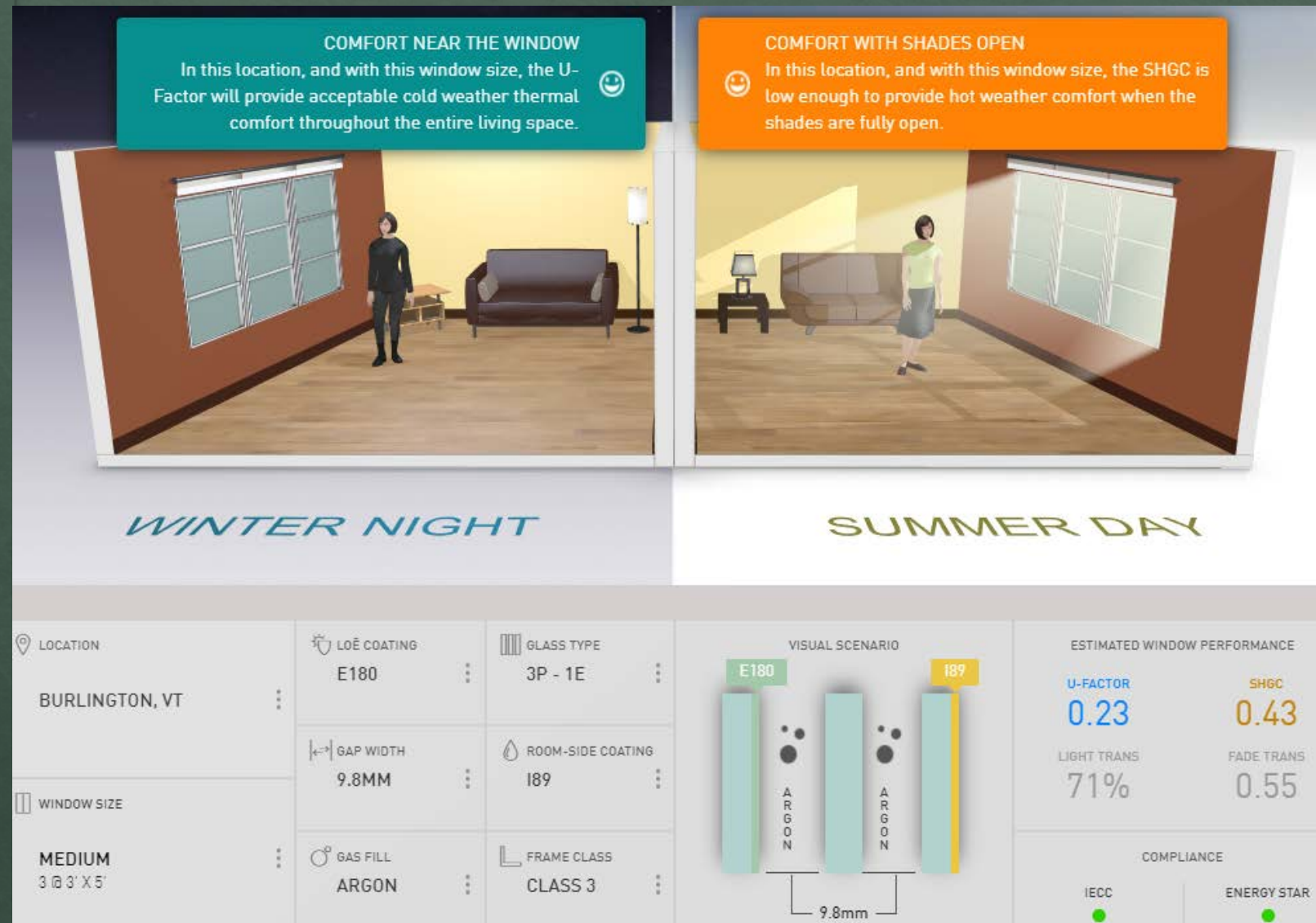


# Cardinal glazing tool

<https://www.cardinalcorp.com/technology/applications/comfort-calculator/>

Tweak a little (in this case to a triple)

...and the blue band disappears



# Designing for comfort



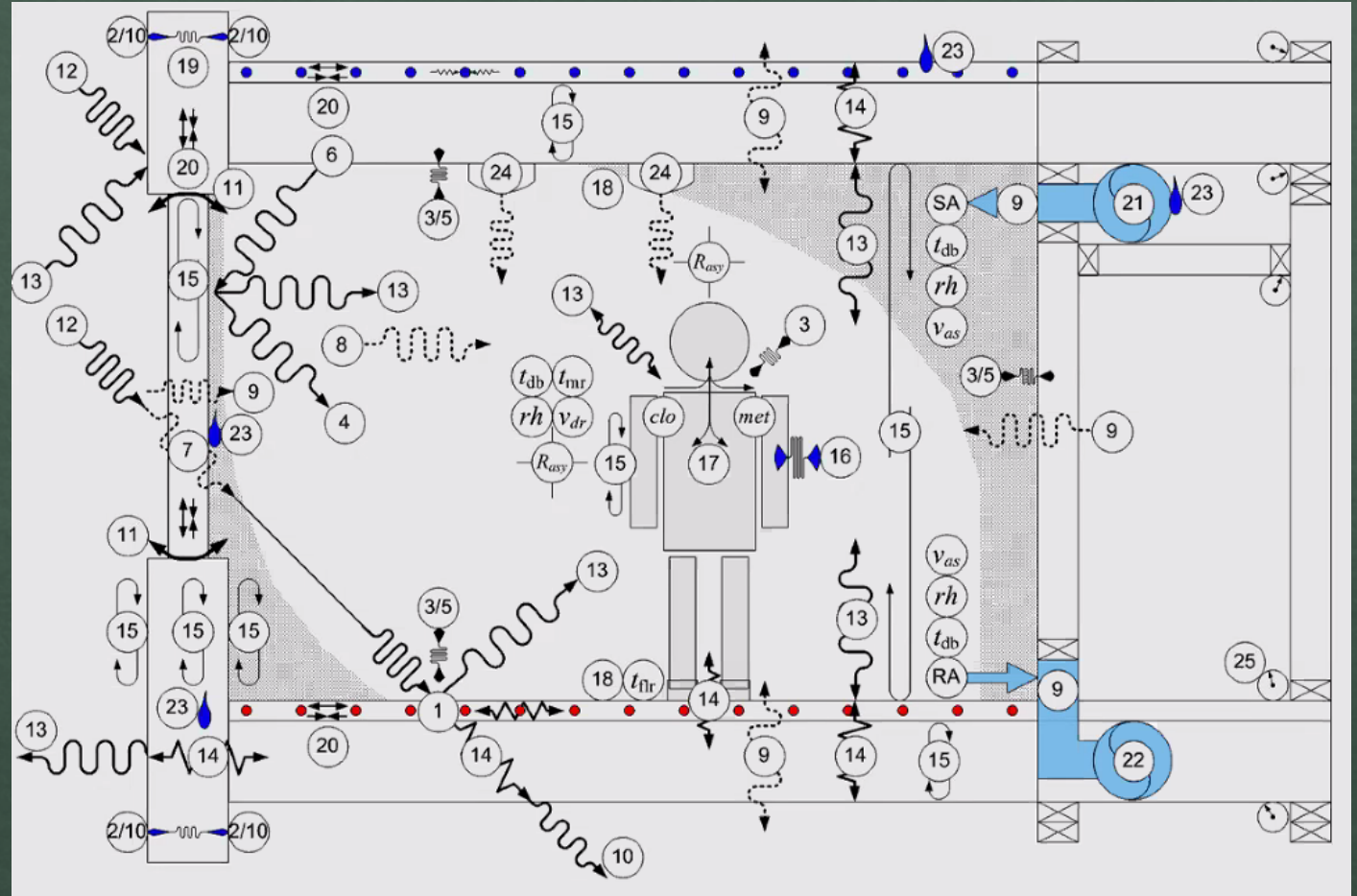
## Remember...

There's a lot going on.

Codes ignore the effects of most of this.

The human body does not.

This would be easier to nail down if it weren't for the people.



Courtesy Robert Bean <http://www.healthyheating.com/>

# Thermal comfort review

- Metabolic rate (met)
  - Clothing insulation (clo)
  - Radiant temperature
  - Air temperature
  - Air speed
  - Humidity
- Personal factors  
you can't reliably influence
- **One-time opportunity** (design)
- Controls can help (but some  
require design/infrastructure)



# Steps (mine, not ASHRAE's)

1. Model. Use free tools to justify large surfaces being as well-insulated as reasonable.
  - Use ASHRAE numbers for metabolic rate and summer/winter clothing insulation
2. Check that the local discomfort guidelines are being met (these are ignored by the software).
3. If (1) or (2) are out of line, point out the comfort hit to client *which a higher thermostat setting may not totally fix*. Think of ways to improve this.



# Costs of compensating?

Assume a moderate winter condition:

- 15' x 20' living room
- Person with typical winter clothing seated 2.5' from a picture window
- 30% relative humidity
- 70F air temperature (10F outside)
- **Code-level** building shell, draft-free (aside from window effects)

How do we make this person comfortable?



# Costs of compensating, cont.

As is:

**X** Does not comply with ASHRAE Standard 55-2017

PMV = -1.36

PPD = 43 %

Sensation = Slightly Cool

SET = 70.0 °F

(and this is with an air temperature of 70F)

# Costs of compensating, cont.

As is:

✗ Does not comply with ASHRAE Standard 55-2017

PMV = -1.67  
Sensation = Cool

PPD = 60 %  
SET = 68.2 °F

(and this is with an air temperature of 70F)

To reach borderline comfort, options include:

Compensator*	Notes
Increase air temp to 82F	Radiant discomfort?
Add thick sweater <u>and</u> increase air temp to 73F	Must wear sweater?
Add a vest <u>and</u> heat floor to 90F	Floor/air discrepancy?



# Costs of compensating, cont.

As is:

✗ Does not comply with ASHRAE Standard 55-2017

PMV = -1.67

PPD = 60 %

Sensation = Cool

SET = 68.2 °F

(and this is with an air temperature of 70F)

Or: Add build a great thermal shell and add a long sleeve t-shirt

# Costs of compensating?

Assume a moderate winter condition:

- 15' x 20' living room
- Person with typical winter clothing seated 2.5' from a picture window
- 30% relative humidity
- 70F air temperature (10F outside)
- **Code-level** building shell, draft-free (aside from window effects)

How do we make this person comfortable?



Grade: D

“The worst you can legally build to”



# My takeaways

- Aim higher than code-level windows
  - Double-pane / ENERGY STAR windows can still cause downdraft discomfort and strongly contribute to radiant discomfort (in addition to condensation and frost)
  - VEIC's old office on a 15F day: 55F glass, 44F frame edges
- Don't put a cold floor or a ground-connected heat sink under your feet
  - Many codes do not require sub-slab insulation for unheated slabs
- Insulate your walls and ceilings
  - Bonus: This goes a long way towards eliminating the pesky local discomfort items



# Choose one

Design for  
comfort

(this does not  
mean code  
minimums)

OR

Set client expectation that 2 or more of the following might be needed to achieve comfort in their dream home:\*

- Additional clothing (e.g. shoes, multiple sweaters)
- Very-warm radiant floor
- Woodstove on
- Avoid sitting near windows
- 75F-plus setpoint



# Beyond ASHRAE

# 1: IEQ

IEQ = Indoor Environmental Quality

$$IEQ = IAQ + ITQ + ILQ + ISQ + IOQ + IVQ$$

where I = Indoor, Q = Quality

and	A =	Air	Much more to comfort than just thermal!
	T =	Thermal	
	L =	Lighting	
	S =	Sound	
	O =	Odor	
	V =	Vibrations	



# 1: IEQ

IEQ = Indoor Envi

Ventilation, ability to operate windows, healthy materials

Insulation, air sealing, windows, well-designed heating and cooling

$$IEQ = IAQ + ITQ + ILQ + ISQ + VOC + MVQ$$

Windows, lighting bulb/fixture design

where I = Indoor, Q = Quality

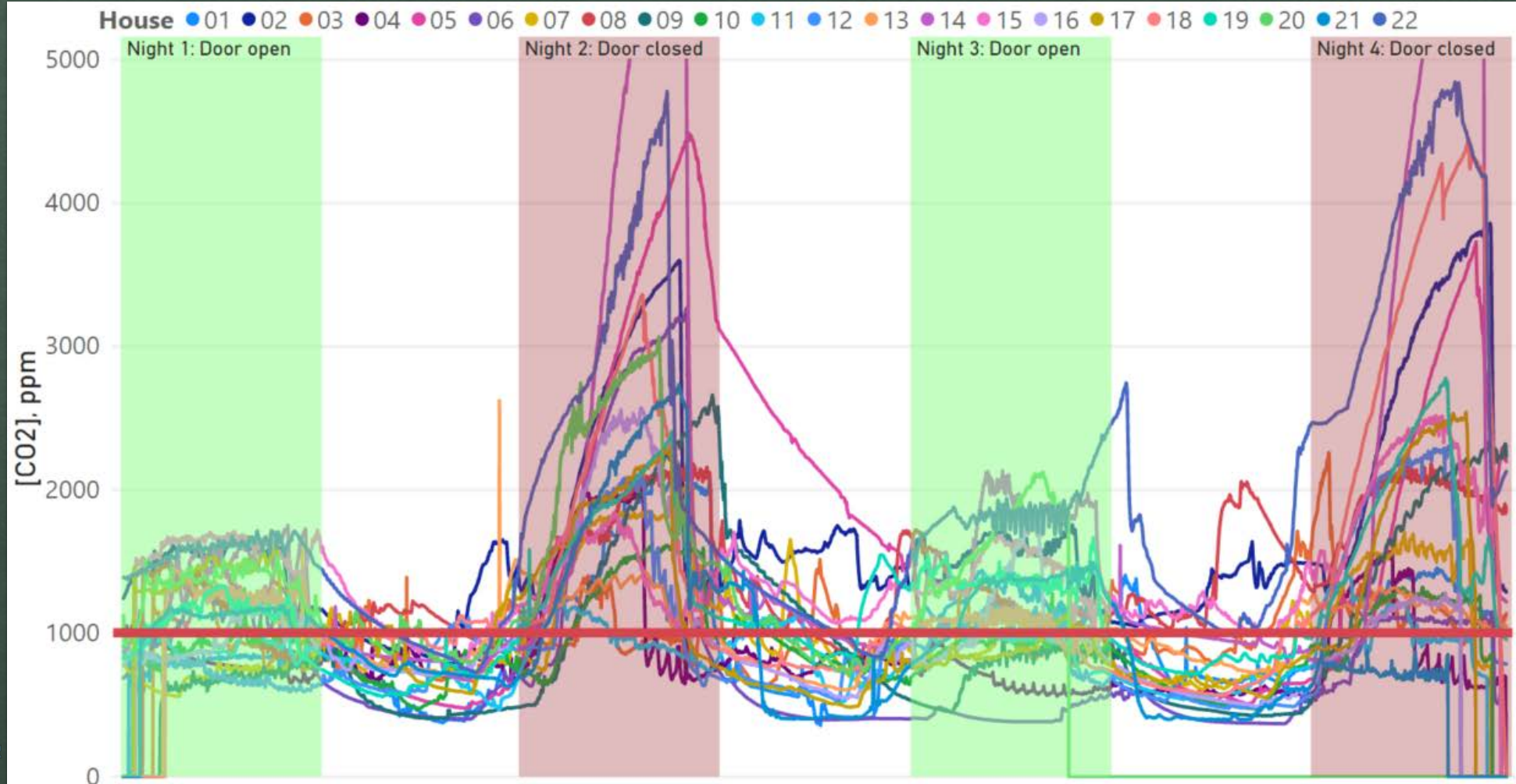
and

A = Air  
T = Thermal  
L = Lighting  
S = Sound  
O = Odor  
V = Vibrations

Insulation, windows, orientation

Ventilation, air sealing (between MF units), healthy materials

# 1. IAQ tangent: Ventilate, right



IAQ &

Ref: Breathe Well, Sleep Well: Cold-Climate Ventilation,

<https://www.efficiencyvermont.com/news-blog/whitepapers/breathe-well-sleep-well-improving-ventilation-in-cold-climate-homes>



## 2: My floor feels cold!

Contact coefficient ( $\text{kCal}/\text{Cm}^2\text{hr}^{0.5}$ ) →

The higher the contact coefficient, the better the material is at *drawing heat out of the feet*

- Concrete 25
- Linoleum 9
- Oak 7
- Pine 4
- Cork 2



# 3: Adaptive comfort

## 1. *Behavioral adaptation*, or **control**

- Putting occupant in the “driver’s seat” can expand comfort zone
- Conscious and unconscious actions that we take to adjust our thermal environment

## 2. *Physiological adaptation*, or **acclimatization**

- Biological changes caused by long-term exposure to conditions

## 3. *Psychological adaptation*, or **expecting/getting used to it**

- Perception/reaction to conditions due to past experience or present/future expectations
- E.g. cool weather *outside* can lead to people being comfortable at a slightly lower range of temperatures *inside*
- James Wise: “anytime you can get anyone in engineering to pay attention to anything that comes out of psychology, it’s a major victory”



### 3: Adaptive comfort → Impacts

We seasonally shift our comfort range

- Wearing shorts on the first 50 degree day of spring vs. first week of November
- 85F feels great outdoors (with shade and a breeze), but usually terrible in a sealed building

“Thermal boredom” (dislike of unchanging conditions) is real

- Donald McIntyre: “It can be argued that achieving a steady optimum temperature is akin to finding the most popular meal at the canteen and then serving it every day.”



## 3: Adaptive comfort → Impacts

Control over conditions expands individual comfort zones

- Give the 20% (ASHRAE 55) of people who are uncomfortable the ability to fine tune their situation

**Naturally ventilated spaces** (where air is supplied/removed without mechanical systems) can **expand comfort zones** (and ASHRAE 55 recognizes it)

- Research by Gail Brager and Richard de Dear found that occupants of mechanically ventilated buildings were twice as sensitive to temperature changes as those in naturally ventilated buildings
- “Such a finding suggests that people in air-conditioned buildings have higher expectations for thermal consistency,” they explain, “and quickly become critical if thermal conditions diverge from these expectations.” In other words, we can become addicted to air conditioning.



## 4: Hedonic response

- “Thermal delight”
- Relies on contrasts
- Example:
  - On a cold day, it feels great to enter an overly warm building
  - As you adjust, it might feel *too* warm after 10 minutes



## 4: Hedonic response

- No: Uniformity
    - Stable temperature
    - Brute-force, sealed environment
    - “Acceptability” for 80% of people (ASHRAE 55)
  - Yes: Variety, Sovereignty
    - Temperatures float in line with natural swings
    - Exploiting air motion
    - Individual control
    - Surface choices (e.g. soft leather on walls, cool terracotta tiles underfoot, the feel of wood grain on your hands)
- } Thermal landscaping



## 5: Room to room variation

- ASHRAE 55 says nothing
- ACCA\* Manual J:
  - 2F ideal
  - 4F allowable

\*Air Conditioning Contractors of America

## 6: More on windows

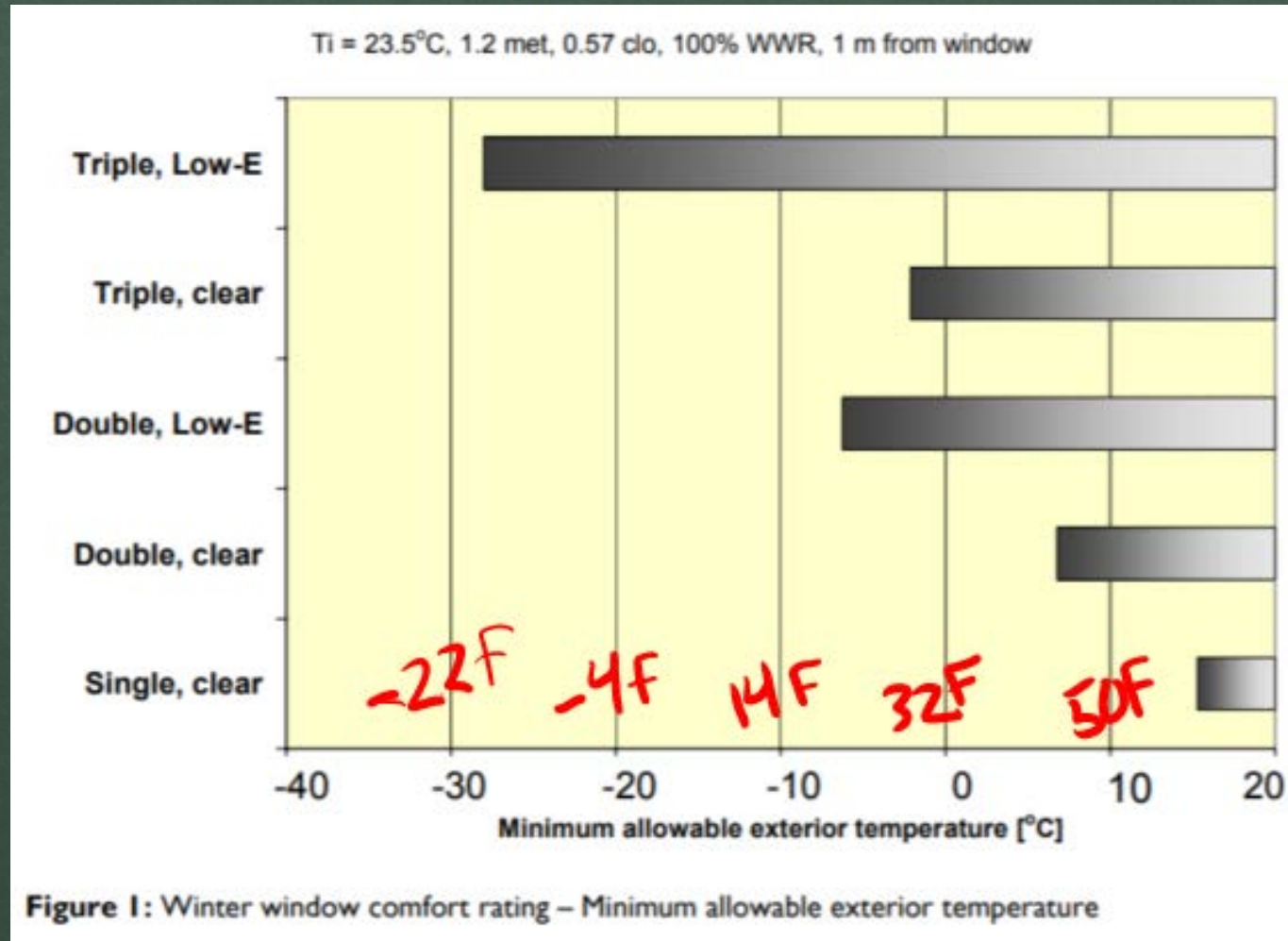
Source: *Window Performance for Human Thermal Comfort*

- Highlights from approx. 200 papers and studies
- Developed “Window Comfort Rating” equation for winter



## 6: More on windows

These results indicate that even low-e double pane windows are **only good to about 20F**



## 7: Less analytical, more human

- Victor Olgyay, 1963
- Design with Climate: A Bioclimatic Approach to Architectural Regionalism

Olgyay's Model

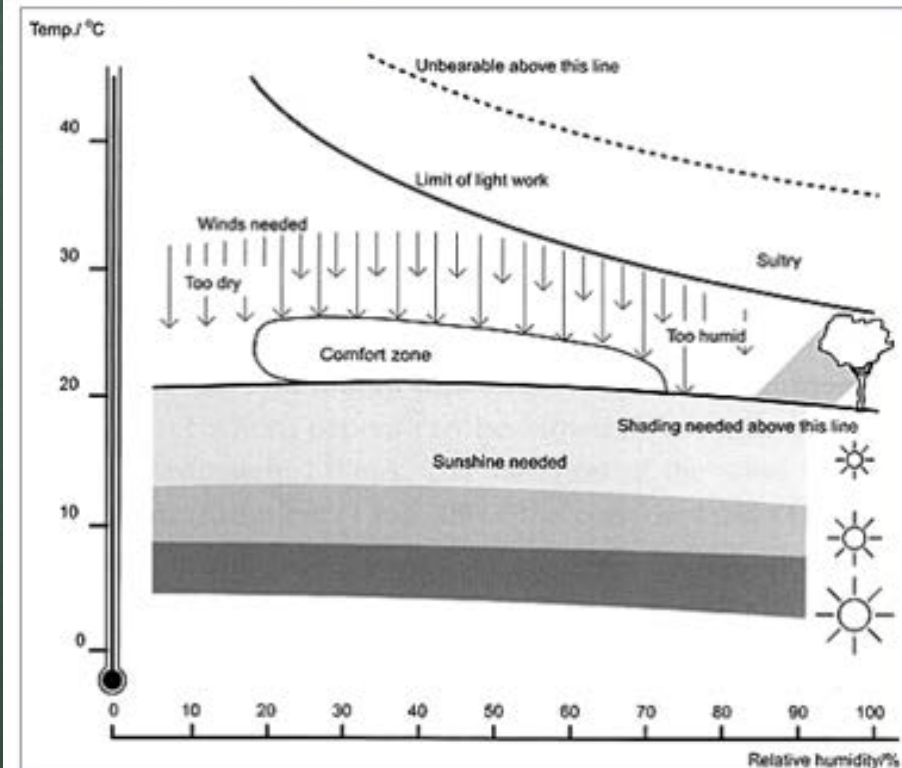


Figure 3. Simplified version of Olgyay's bioclimatic chart of thermal comfort zones.

From Stay Cool: A Design Guide for the Built Environment in Hot Climates  
by Holger Koch-Nielsen (James



# Pulling it all together

# Thermal comfort review

- Metabolic rate (met)
  - Clothing insulation (clo)
  - Radiant temperature
  - Air temperature
  - Air speed
  - Humidity
- Personal factors  
you can't reliably influence
- **One-time opportunity** (design)
- Controls can help (but some  
require design/infrastructure)

**Which one(s) are *typically* directly controlled in your home?**  
**Which can be easily tweaked *after* something's built?**



# Code, plus one program's approach...

Requirement	Energy code*	EVT Base	EVT High Performance
Wall insulation (AG and band)	R-20 cavity	<b>R-26**</b> Min R-5 continuous	<b>R-40</b>
Under slab insulation	R-15 heated R-0 unheated	R-15 heated R-0 unheated	<b>R-30 heated or on grade</b> <b>R-20 unheated or below grade</b>
Insulation installation	No req't	Grade I	Grade I
Windows, u-factor	0.28 max	0.28 max	<b>0.21 max</b>
Air leakage, max	3.0 ACH50	2.0 ACH50	1.0 ACH50
Ventilation	EOV allowed	<b>High-efficiency balanced ventilation</b>	<b>High-efficiency balanced ventilation</b>

# My take on all of this

## Design:

- AGW continuous insulation, 2" min.
- Triple-pane windows
- Airtight, 1.0 ACH50 max
- Floor / slab insulation
- 24/7 high efficiency ventilation (don't blow cold air), quiet design (don't be loud), MERV 13 filtration

## And sell its IEQ:\*

- Thermal comfort
- Quietness
- Fresh air-ness
- Natural light (without downdraft or radiant discomfort)
- Sense of wellbeing



# Listen to both sides

Thinking	Feeling
<ul style="list-style-type: none"><li>• Continuous insulation in walls</li><li>• Invest in great windows</li><li>• Air seal really well</li><li>• Insulate all slabs</li><li>• Ventilate with intent</li><li>• Choose floor material wisely</li></ul>	<ul style="list-style-type: none"><li>• Give control (zones, windows, etc.)</li><li>• Give variety* and let occupant adjust via clothing, activity</li><li>• Choose materials with intent</li></ul>

\*E.g. make temperature swings (seasonal, time of day) okay / purposeful





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# Disclaimer

- This presentation is not intended to be a comprehensive program covering all aspects of this topic.
- All are participants are encouraged to read and follow applicable standards, codes and regulations related to this topic.
- The views and opinions following are the presenter's opinions and not necessarily the official position of the Maine IAQ Council, IAQnet LLC, or Healthy Indoors.



# More on windows and solar radiation

Notes from: *Window Performance for Human Thermal Comfort: Final Report to the National Fenestration Rating Council*, Center for the Built Environment & ARUP, Feb 2006,  
<https://escholarship.org/uc/item/6rp85170>

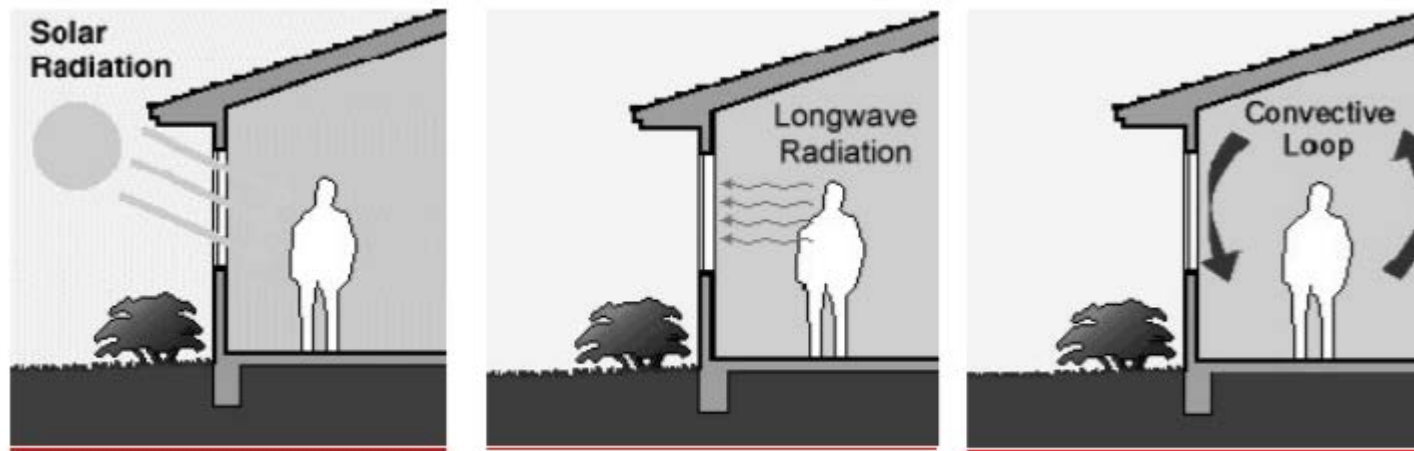
- “With respect to solar radiation, we considered only diffuse radiation based on notion that direct sun falling on the body will cause discomfort in all but the coolest environments and that some action will be taken by the occupant to mitigate direct sun. Diffuse solar still has a significant effect on comfort, though significantly less than direct solar”
- “Absorbed radiation influences the temperature of the glass; the inside surface of heat absorbing glass can routinely reach temperatures above 120°F (50°C) in summer conditions, raising MRT by as much as 15°F (8°C). Transmitted radiation often causes discomfort if it falls directly on the occupant. A person sitting near a window in direct solar radiation can experience heat gain equivalent to a 20°F (11°C) (Arens et al. 1986) rise in mean radiant temperature. These radiant heating and cooling effects act on the occupant’s body asymmetrically, causing some parts of the body to be considerably cooler or warmer than a uniform model like MRT can describe. Models need to consider the effect on local skin temperature in order to be sensitive to discomfort caused by windows.”



# More on windows and solar radiation, cont.

A window influences thermal comfort in three ways (Figure 2):

- long-wave radiation from the warm or cold interior glass surface
- transmitted solar radiation
- induced air motion (convective drafts) caused by a difference between the glass surface temperature and the adjacent air temperature



**Figure 2:** Window impacts on thermal comfort: solar radiation, long-wave radiation, convective drafts